

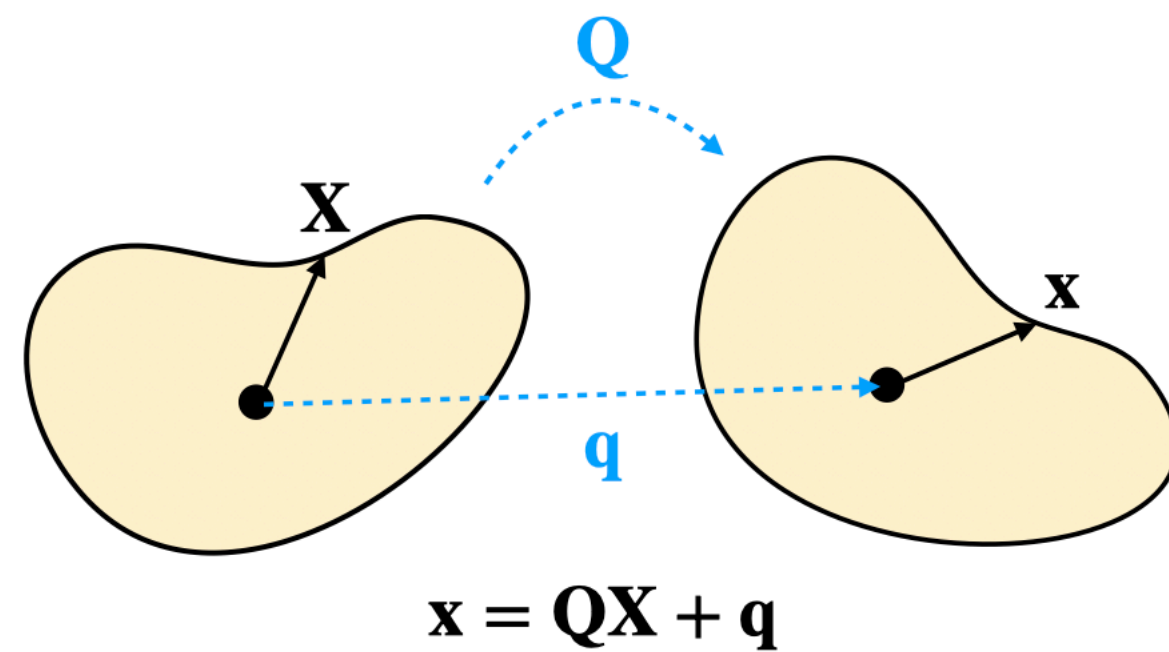
**Instructor: Minchen Li**



# **Lec 13: Codimensional Solids**

**15-763: Physics-based Animation of Solids and Fluids (S25)**

# Recap: Rigid Body Dynamics



Reduced order DOF:

$$\mathbf{x} = \mathbf{Q}\mathbf{X} + \mathbf{q} \in \mathbb{R}^3$$

$\Leftrightarrow$

$$\begin{aligned} \mathbf{x} &= \bar{\mathbf{X}}\mathbf{Q} + \bar{\mathbf{S}}\mathbf{q} \in \mathbb{R}^{3n} \\ \mathbf{Q} &\in \mathbb{R}^{9m}, \quad \bar{\mathbf{X}} \in \mathbb{R}^{3n \times 9m} \\ \mathbf{q} &\in \mathbb{R}^{3m}, \quad \bar{\mathbf{S}} \in \mathbb{R}^{3n \times 3m} \end{aligned}$$

Reduced order dynamics (from subspace optimization):

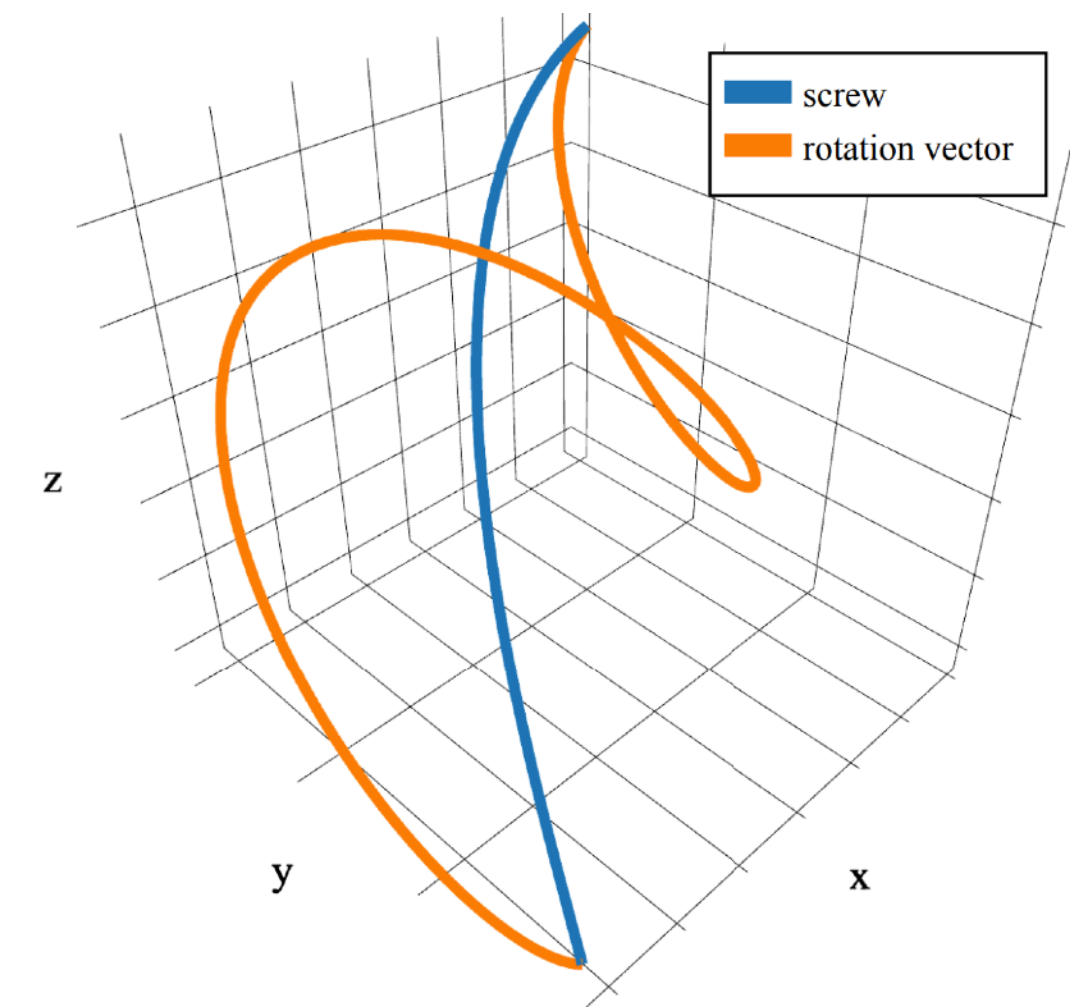
$$\min_{\theta, q} \frac{1}{2} \|\bar{\mathbf{X}}\mathcal{R}(\theta) + \bar{\mathbf{S}}\mathbf{q} - \tilde{\mathbf{x}}^n\|_M^2 + h^2 \sum P(\bar{\mathbf{X}}\mathcal{R}(\theta) + \bar{\mathbf{S}}\mathbf{q})$$

Rodrigues' Rotation Formula:

$$\mathcal{R}(\theta) = \text{Id} + \sin(\|\theta\|) \left[ \frac{\theta}{\|\theta\|} \right] + (1 - \cos(\|\theta\|)) \left[ \frac{\theta}{\|\theta\|} \right]^2$$

Just include IPC energies here

CCD is on nonlinear trajectories:



Very expensive!

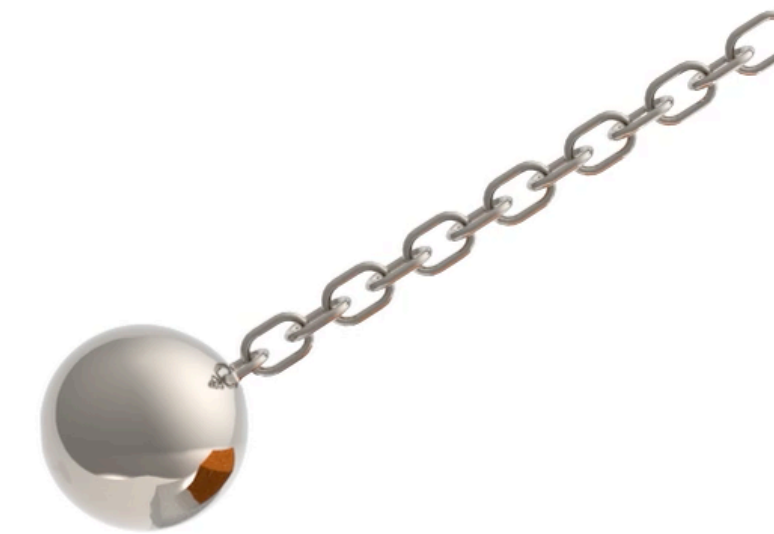


# Recap: Affine Body Dynamics (ABD)

Reduced order dynamics with penalty method:

$$\min_{Q,q} \frac{1}{2} \|\bar{X}Q + \bar{S}q - \tilde{x}^n\|_M^2 + h^2 \sum P(\bar{X}Q + \bar{S}q)$$

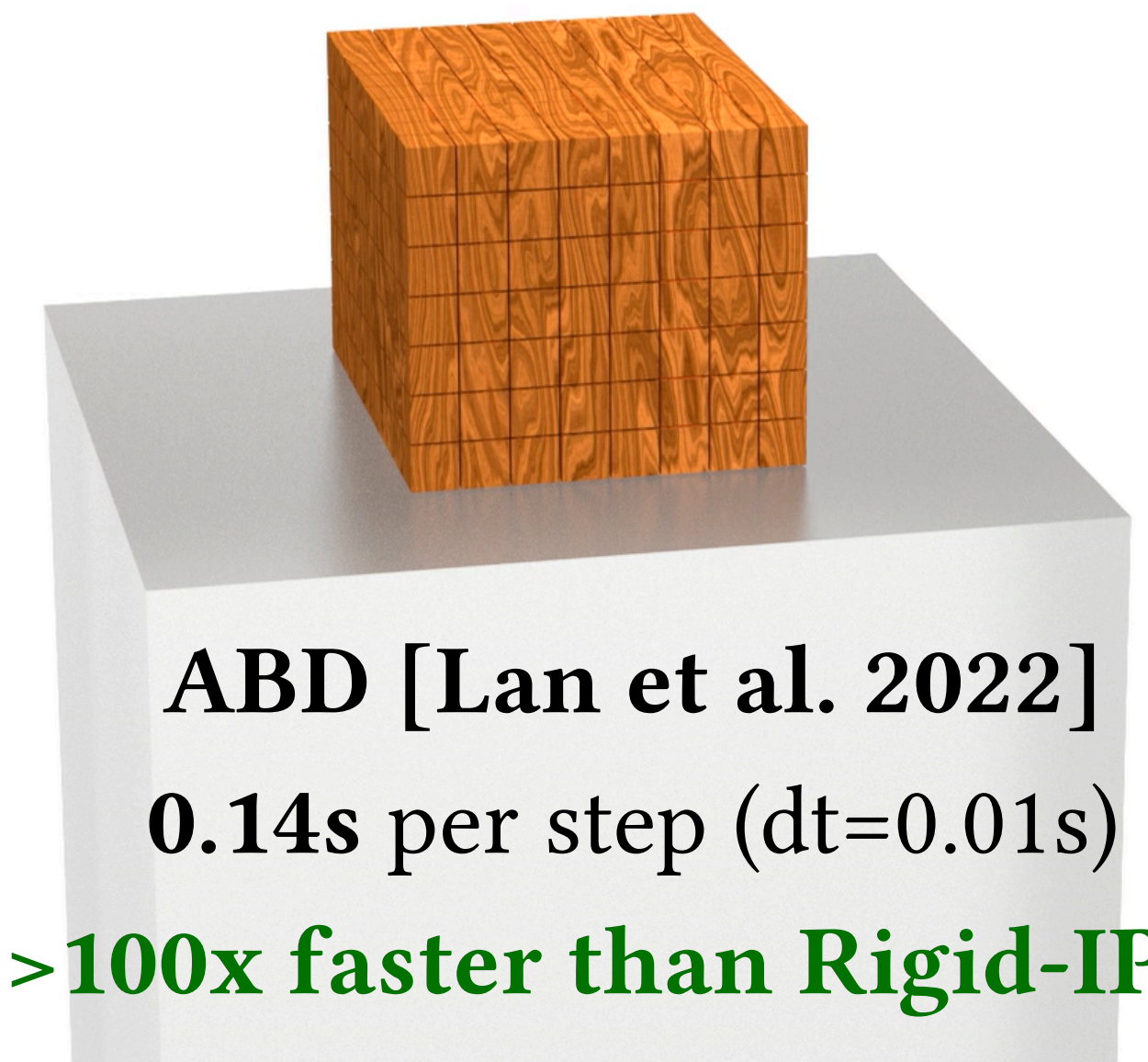
Use elasticity with large  
Young's modulus



12 DOF per body, still significantly reduced

$x = \bar{X}Q + \bar{S}q$  is linear w.r.t. both  $Q$  and  $q$  -> linear CCD

A stiff  $\Psi$  won't make the problem harder with stiff IPC energies



ABD [Lan et al. 2022]

0.14s per step (dt=0.01s)

>100x faster than Rigid-IPC,

# Recap: ABD in Another Perspective

## Affine Deformation Modes

$$\mathbf{x} = \begin{bmatrix} a & b \\ c & d \end{bmatrix} \mathbf{X} + \begin{bmatrix} e \\ f \end{bmatrix}$$

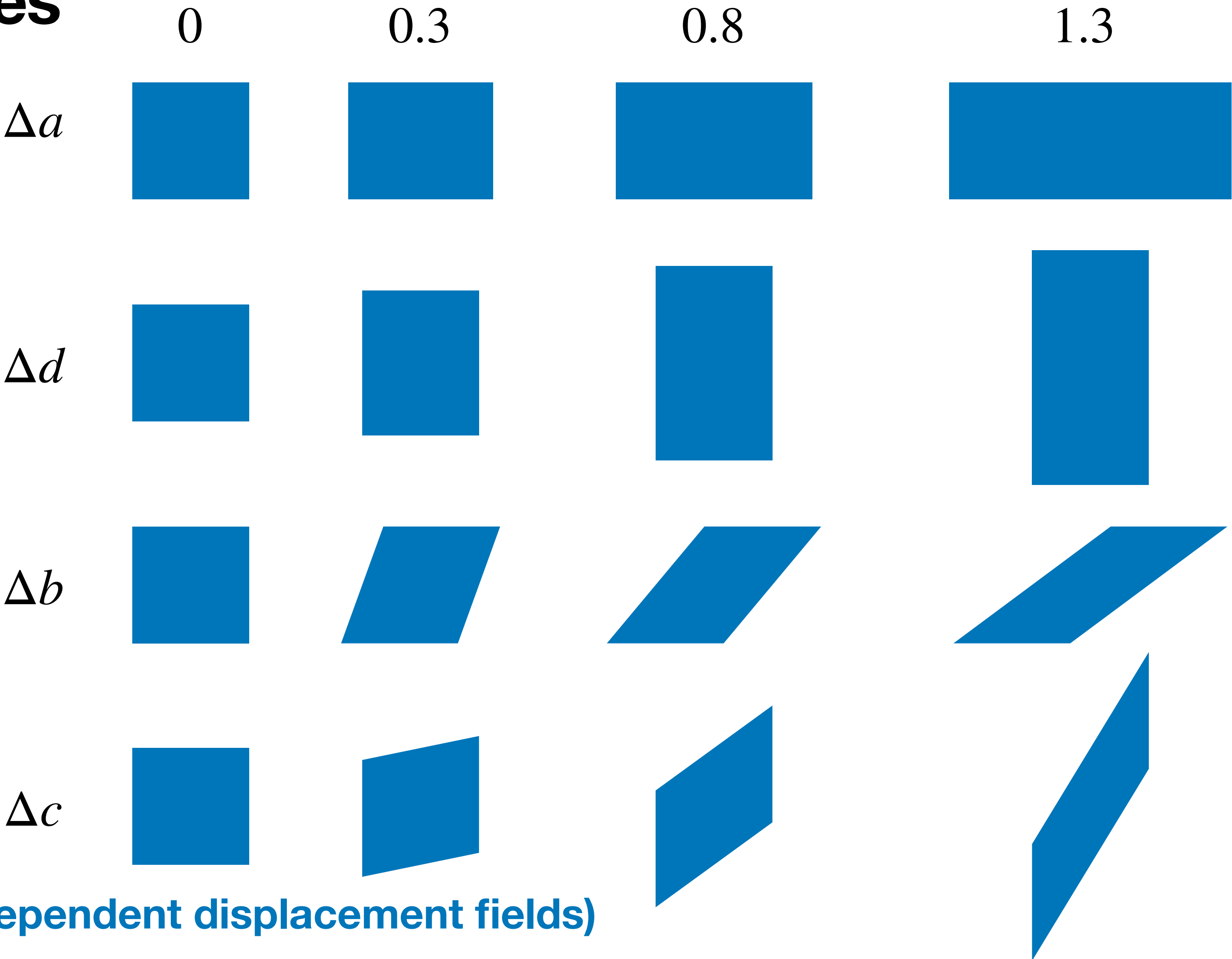
DOF:  $a, b, c, d, e, f$



$\mathbf{x}$ :

$$\mathbf{x} = A \begin{bmatrix} a \\ b \\ c \\ d \\ e \\ f \end{bmatrix} = aA_1 + bA_2 + \dots$$

Deformation modes (linearly independent displacement fields)





# Recap: Reduced Order Model

## Linear Modal Analysis

Assume linear elasticity problem:  $M\ddot{u} + Ku = f$  s.t.  $Sx = 0$  (Dirichlet BC)

**Intuition: Meaningful deformation modes are those don't generate large forces**

Can solve the generalized Eigenvalue problem to find them:  $\bar{K}y = \lambda\bar{M}y$

(where  $\bar{K}$  and  $\bar{M}$  do not account for BC nodes)

(Take the Eigenvectors with smallest Eigenvalues as modes.)

Let  $u = x - X = Uz$ , where  $z \in \mathbb{R}^k$  are the reduced DOF,  $U \in \mathbb{R}^{3n \times k}$  formed by the Eigenvectors,

Plugging in  $M\ddot{u} + Ku = f$ , ignoring BCs for now:  $\ddot{z} + \Lambda z = U^T f$  **Diagonal system! Super fast!**

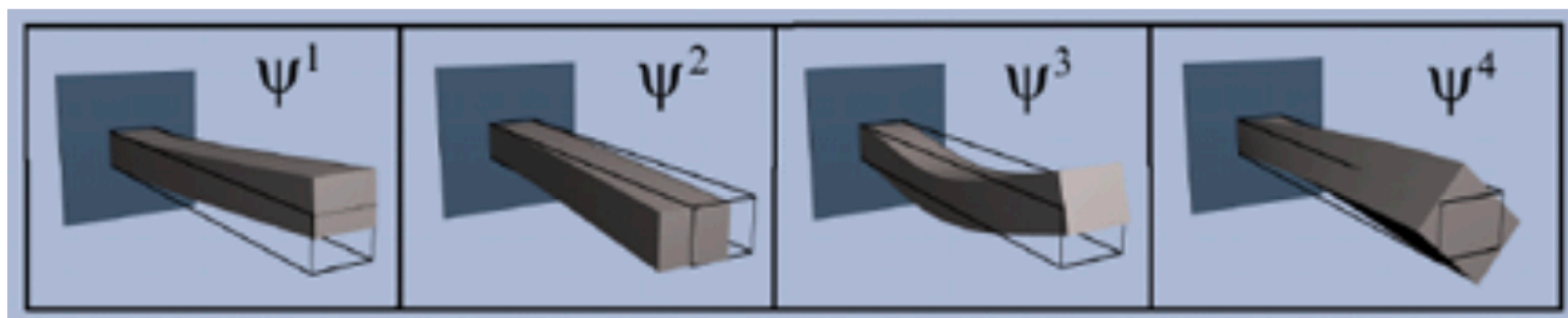


Figure 2: Linear modes for a cantilever beam.

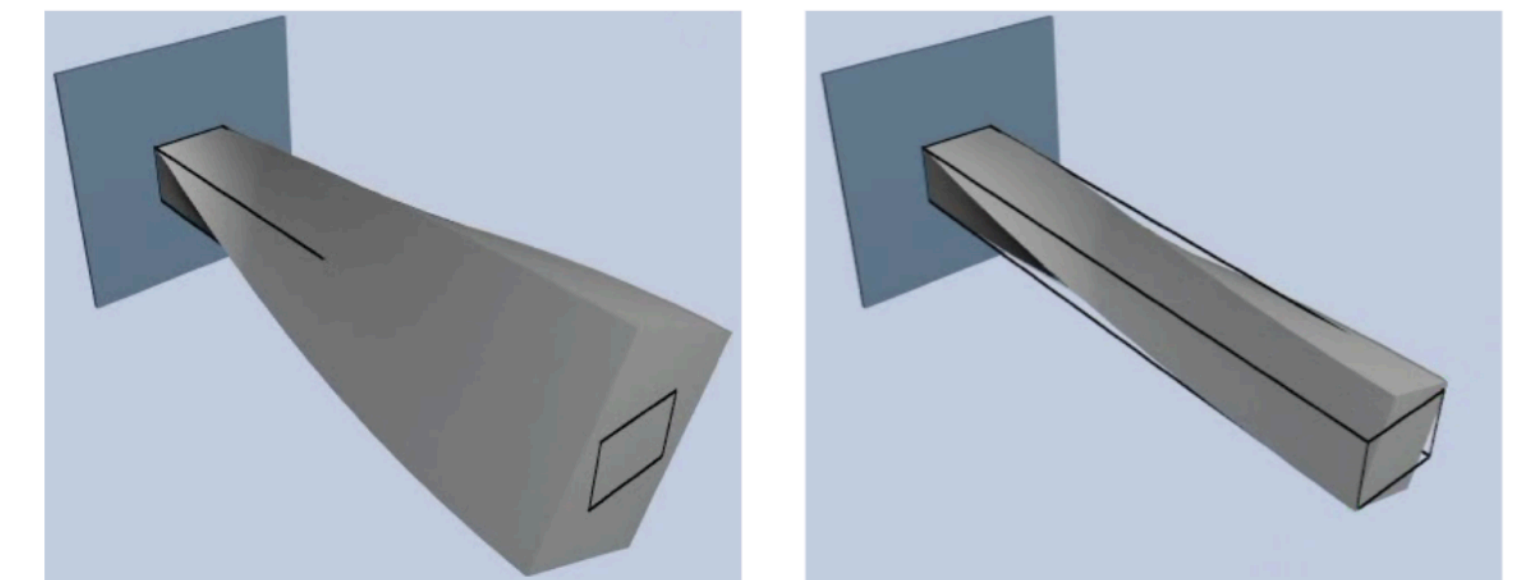


Figure 3: Model reduction applied to a linear and nonlinear system.

# Recap: Reduced Order Model

## Nonlinear Elasticity

Plugging in  $u = Uz$ :  $\min_z \frac{1}{2} \|X + Uz - \tilde{x}^n\|_M^2 + h^2 \sum P(X + Uz)$

**Gradient:**  $U^T M(X + Uz - \tilde{x}^n) + h^2 \sum U^T \nabla P(X + Uz)$

**Hessian:**  $U^T MU + h^2 \sum U^T \nabla^2 P(X + Uz) U$

Can use numerical integration to approximate Gradient and Hessian, minimizing the number of quadratures [An et al. 2008]

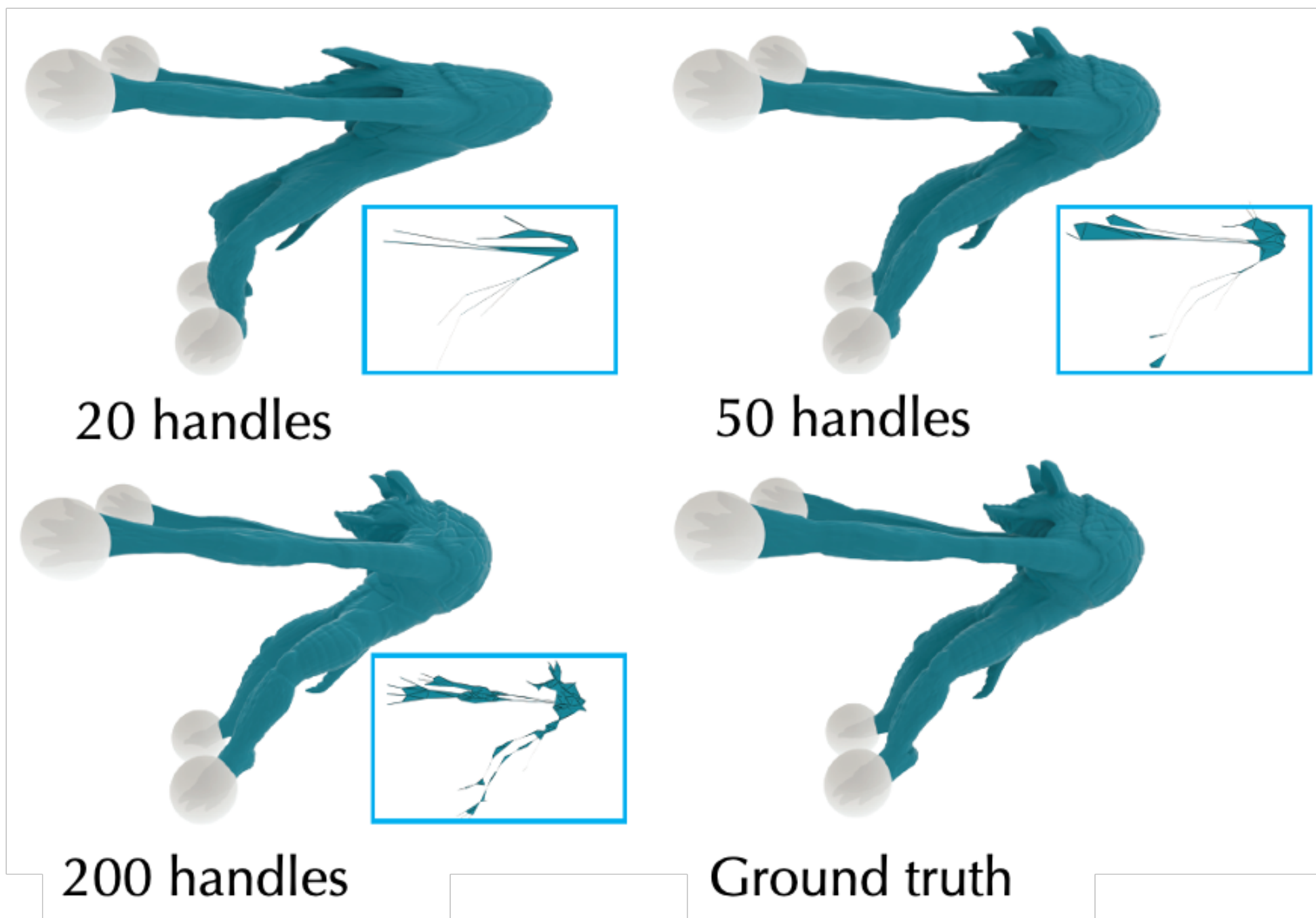
To design better deformation modes:

- Use locally supported modes for sparsity, e.g. Medial Axis Mesh [Lan et al. 2021]
- Use simulated poses/deformed configurations as data, and perform PCA
- Use modal derivatives to construct a quadratic function  $u = f(z)$  [\*]
- Use neural networks to learn  $u = f(z)$

\*Jernej Barbic, Doug James. Real-Time Subspace Integration for St.Venant-Kirchhoff Deformable Models. SIGGRAPH 2005.

# Results: Reduced Simulation of Deformable Solids

## Medial IPC [Lan et al. 2021]



**Puffer Ball x 1**  
*36× speedup*

# of Handles: 1624  
# of Elements: 625k



*Medial IPC*



*Full IPC*



# Topics Today:

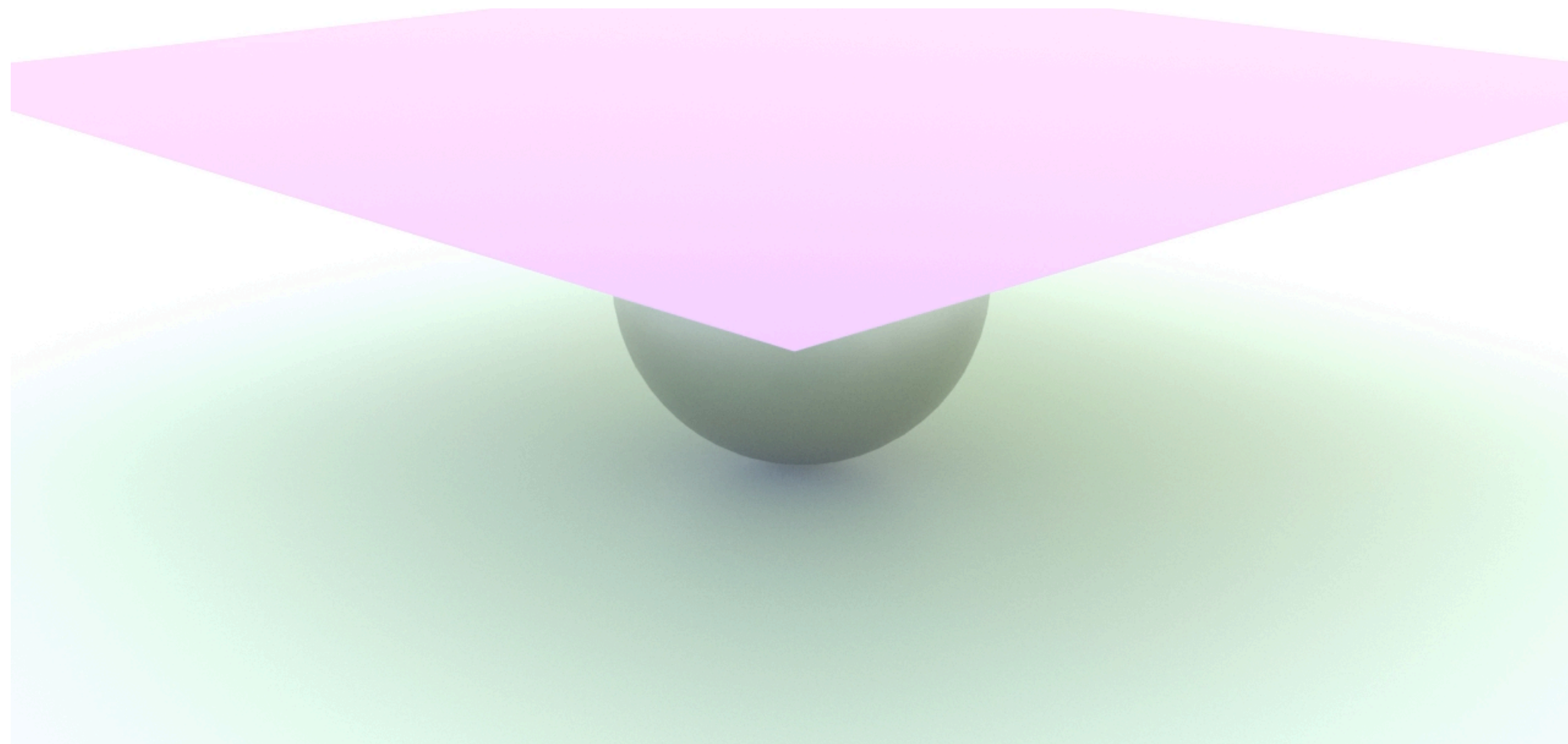
- **Thin Shells**
  - ▶ **Stretching and Bending**
  - ▶ **Strain Limiting and Thickness**
- **Rods and Particles**

# Topics Today:

- **Thin Shells**
  - ▶ **Stretching and Bending**
  - ▶ Strain Limiting and Thickness
- Rods and Particles

# Simulating Thin Objects

## Using Volumetric Meshes?



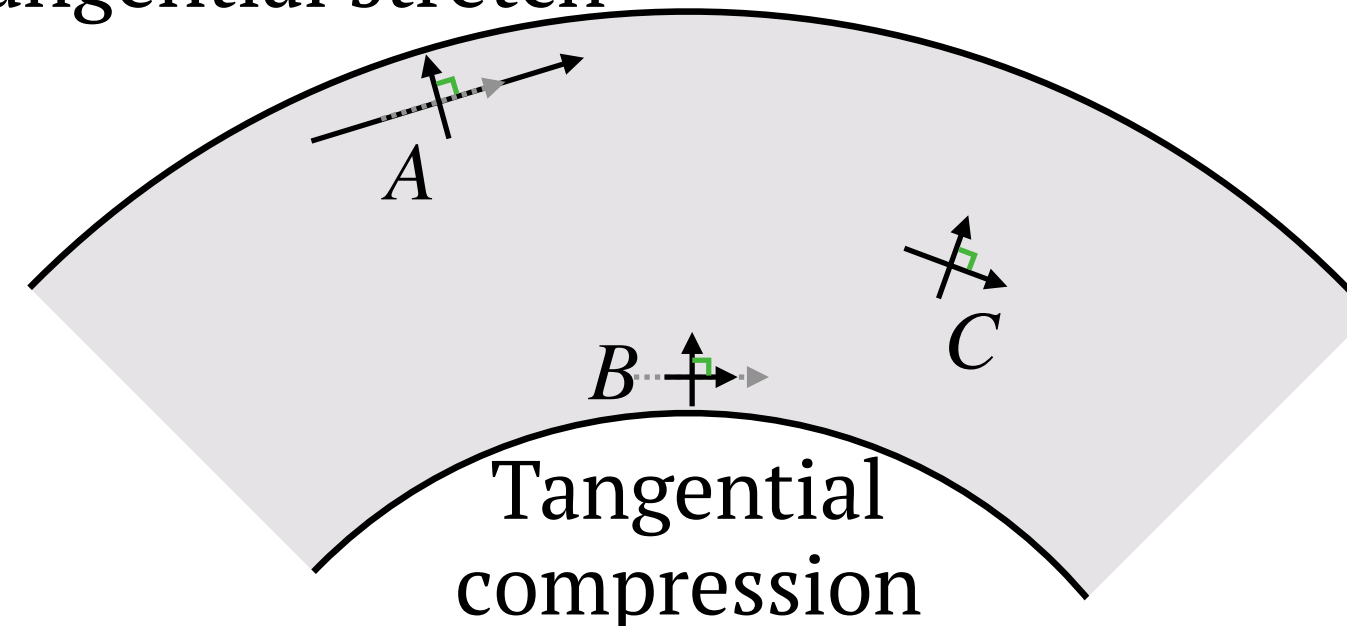
Ill-conditioning:



Shear locking issue  
(linear shape functions):

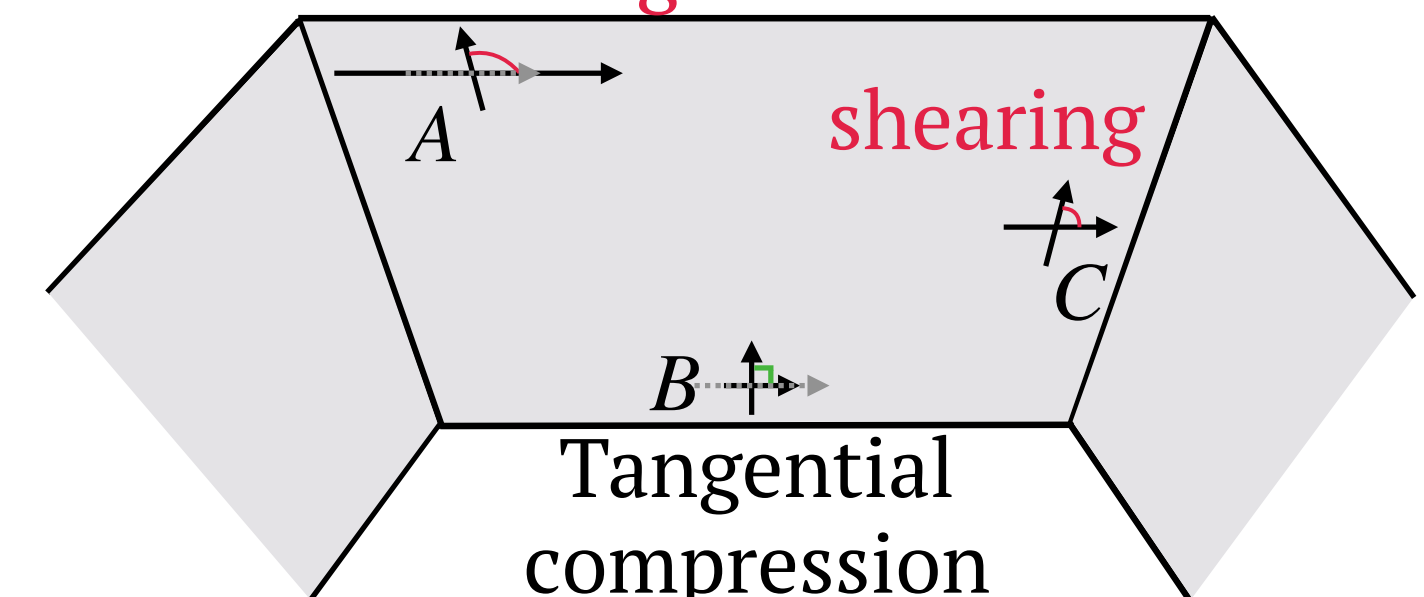
Higher-order shape  
functions are expensive

Tangential stretch



(a) Ideal Setting

Tangential stretch  
and **shearing**



(b) Coarse Tessellation with Linear Basis



# Simulating Thin Shells using Surface Meshes

## Tangent Space Elasticity

Thickness is tiny and barely visible -> can ignore it: 

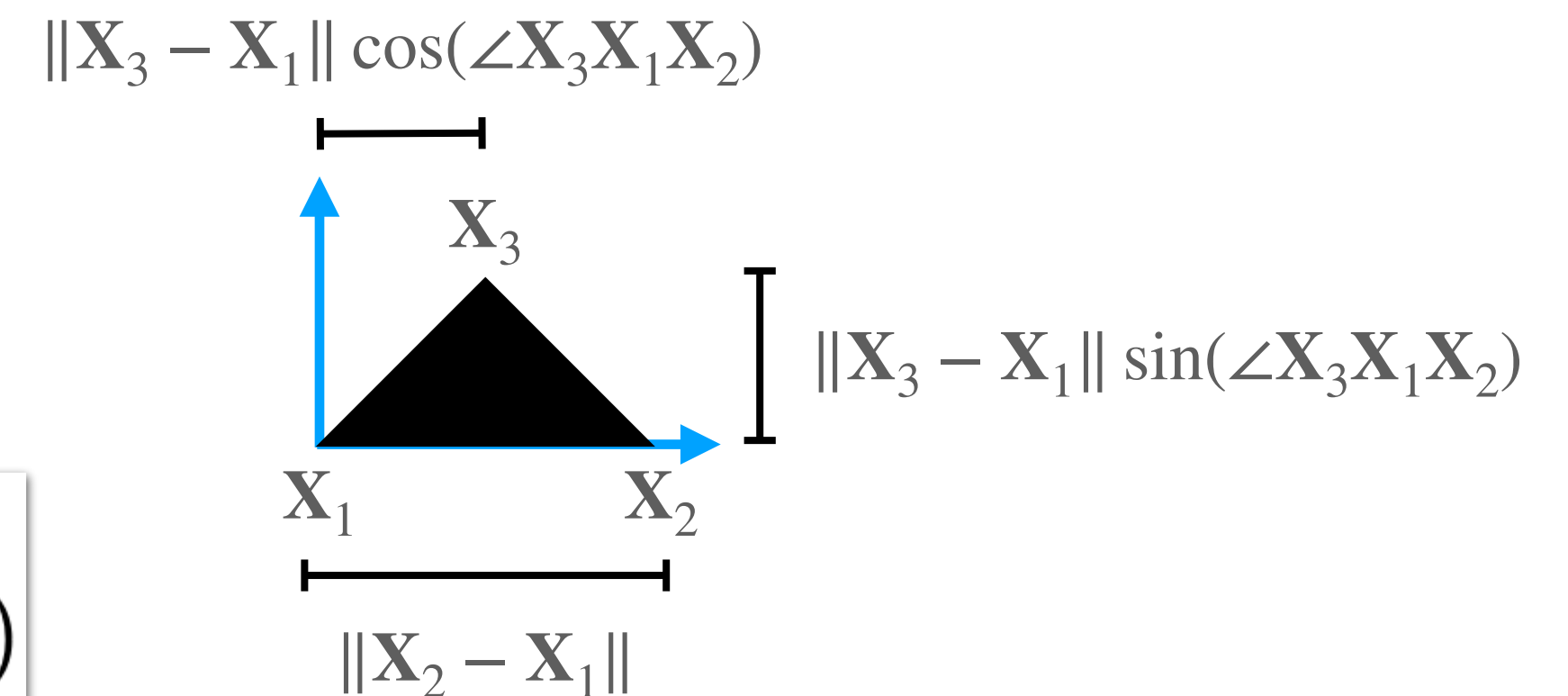
Use surface meshes, e.g. triangle meshes, to simulate thin shells, e.g. cloth, paper, etc

$$F = \begin{bmatrix} x_2 - x_1, x_3 - x_1 \end{bmatrix} \begin{bmatrix} \|X_2 - X_1\| & \frac{(X_3 - X_1) \cdot (X_2 - X_1)}{\|X_2 - X_1\|} \\ 0 & \frac{\|(X_3 - X_1) \times (X_2 - X_1)\|}{\|X_2 - X_1\|} \end{bmatrix}^{-1} \in R^{3 \times 2}$$

$$\mathbf{F} = \mathbf{U}\mathbf{\Sigma}\mathbf{V}^T \quad \hat{\Psi}_{\text{NH}}(\mathbf{\Sigma}) = \frac{\mu}{2} \left( \sum_i^d \sigma_i^2 - d \right) - \mu \ln(J) + \frac{\lambda}{2} \ln^2(J)$$

Lame parameter  $\lambda$  is computed differently here for shells as the thickness approaches zero:

$$\lambda = \frac{E\nu}{1 - \nu^2} \quad (\text{Plane stress approximation}) \quad \lambda = \frac{E\nu}{(1 + \nu)(1 - 2\nu)} \quad (\text{Volumetric})$$



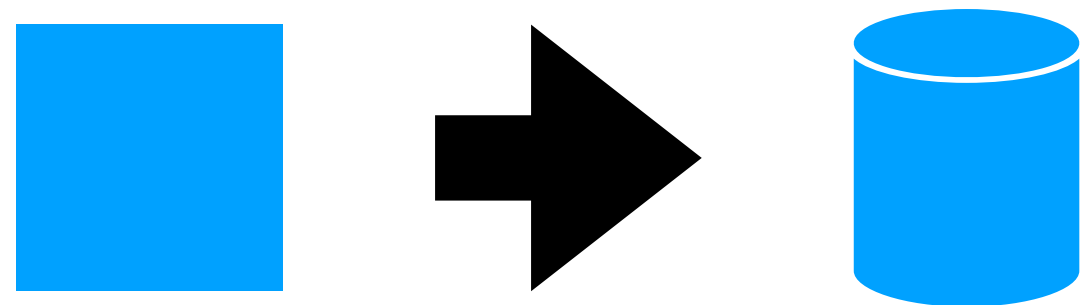
$$\sum_i V_i \Psi(\mathbf{F}_i) \quad V_i = \overset{\text{Thickness}}{\xi} \underset{\text{Triangle area}}{A_i}$$

\*May also use The First Fundamental Form to compute  $\Psi$ .

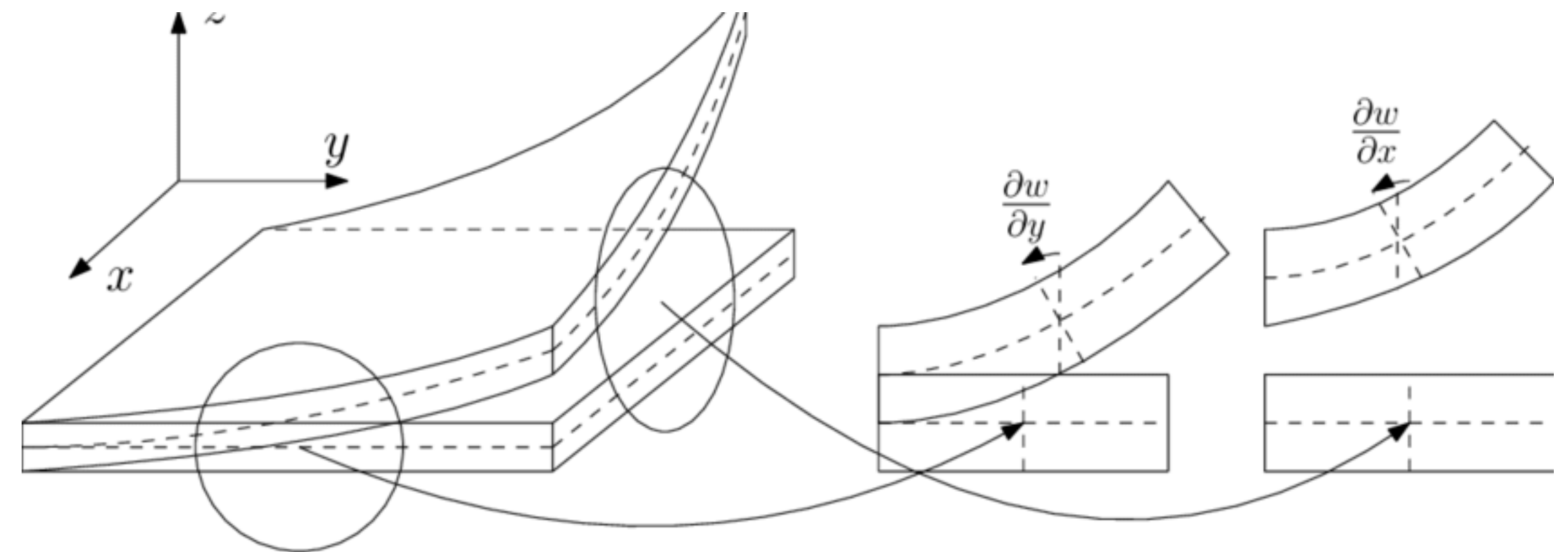
# Simulating Thin Shells using Surface Meshes

## Bending

With only tangent space elasticity, no force under isometric deformation:



Need to additionally model bending resistance!



### The Kirchhoff-Love Model

Deriving strain energies for bending from continuum mechanics, based on assumptions:

- straight lines normal to the mid-surface remain straight and normal to the mid-surface after deformation
- the thickness of the plate does not change during a deformation.

# Simulating Thin Shells using Surface Meshes

## Discrete Shell — Hinge Bending Model [Grinspun et al. 2003]

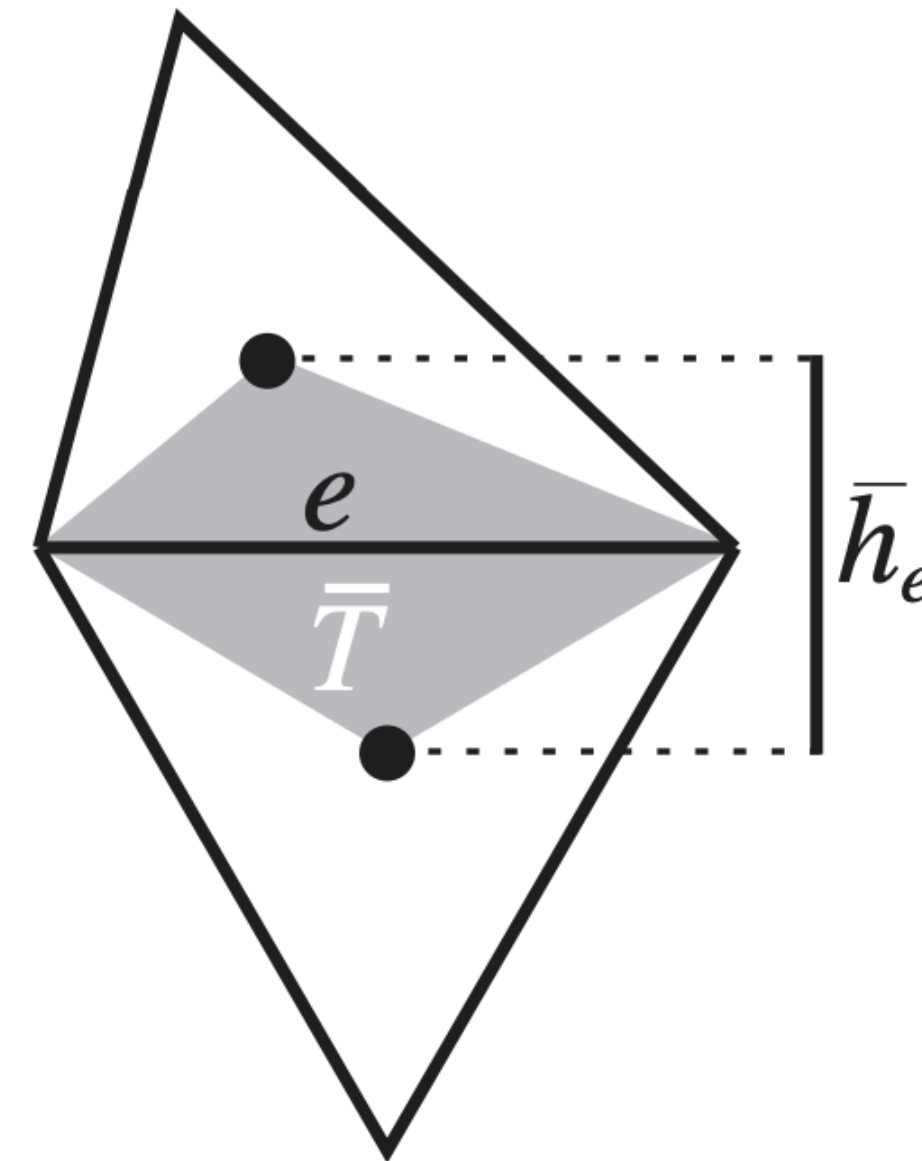
Model the strain energy for bending directly as a penalty of mean curvature changes

$$\int_{\bar{\Omega}} (H \circ \varphi - \bar{H})^2 d\bar{A}$$

Discretizing onto edges of triangle meshes:

$$W_B(\mathbf{x}) = \sum_e (\theta_e - \bar{\theta}_e)^2 \|\bar{\mathbf{e}}\| / \bar{h}_e$$

$\theta$  is the dihedral angle, which  
can be computed with  $\arctan()$



Adding stiffness and reparameterize:

$$\Psi_{\text{bend}}(x) = \sum_i k \frac{3\|\bar{\mathbf{e}}_i\|^2}{\bar{A}_i} (\theta_i - \bar{\theta}_i)^2$$

$$k = \frac{E\xi^3}{24(1 - \nu^2)}$$



# Simulating Thin Shells using Surface Meshes

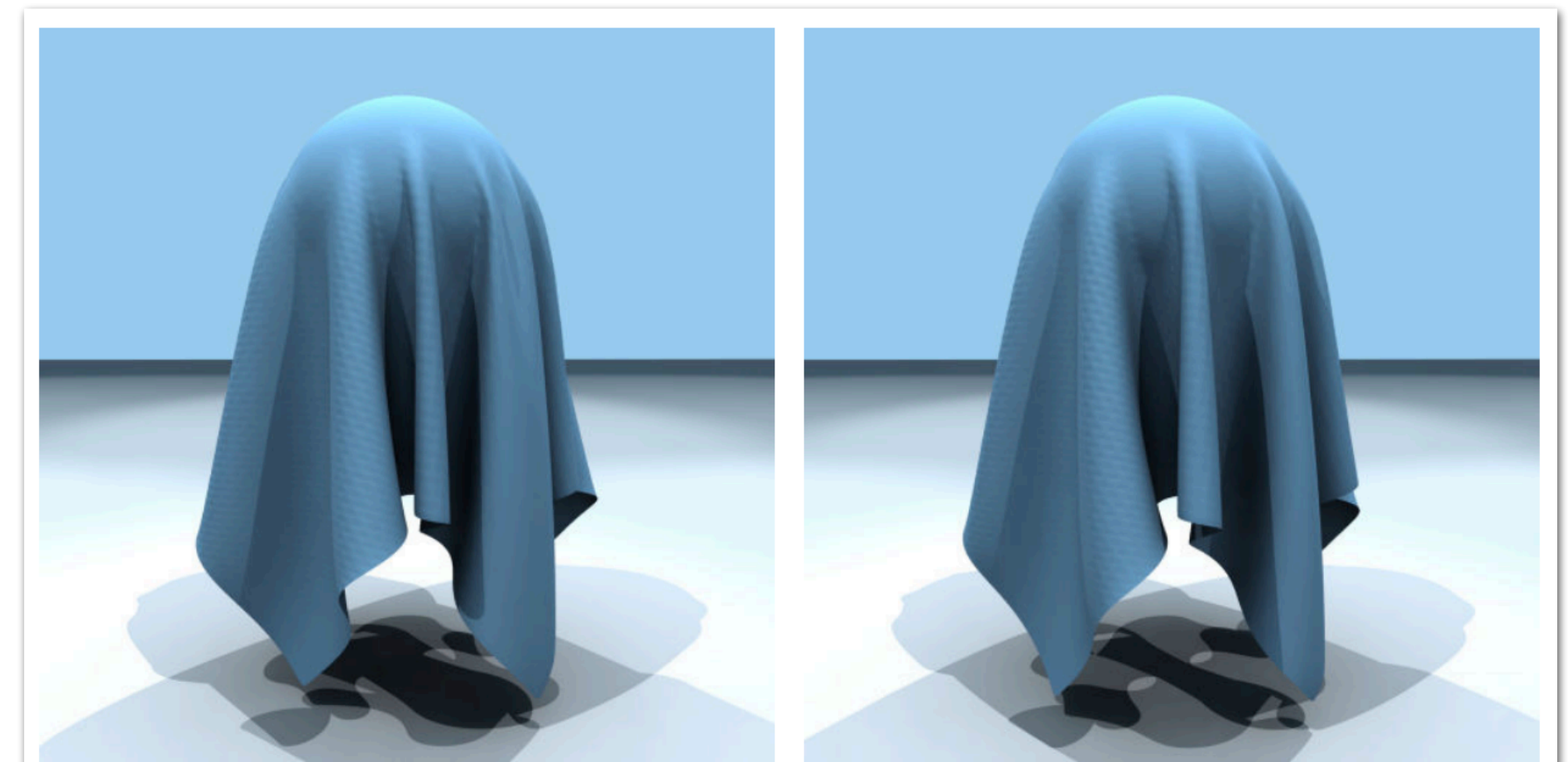
## Cubic and Quadratic Bending Energies

Garg et al. [2007]:

For isometric deformation,  
A bending energy can be formulated as  
a cubic polynomial of  $\chi$

Bergou et al. [2006]:

For isometric deformation of plates (flat rest shapes),  
A bending energy can be formulated as  
a quadratic polynomial of  $\chi$



**Figure 1:** *Final rest state of a cloth draped over a sphere, for (left) the proposed isometric bending model and (right) the widely-adopted nonlinear hinge model.*

# Topics Today:

- **Thin Shells**
  - ▶ Stretching and Bending
  - ▶ **Strain Limiting and Thickness**
- Rods and Particles

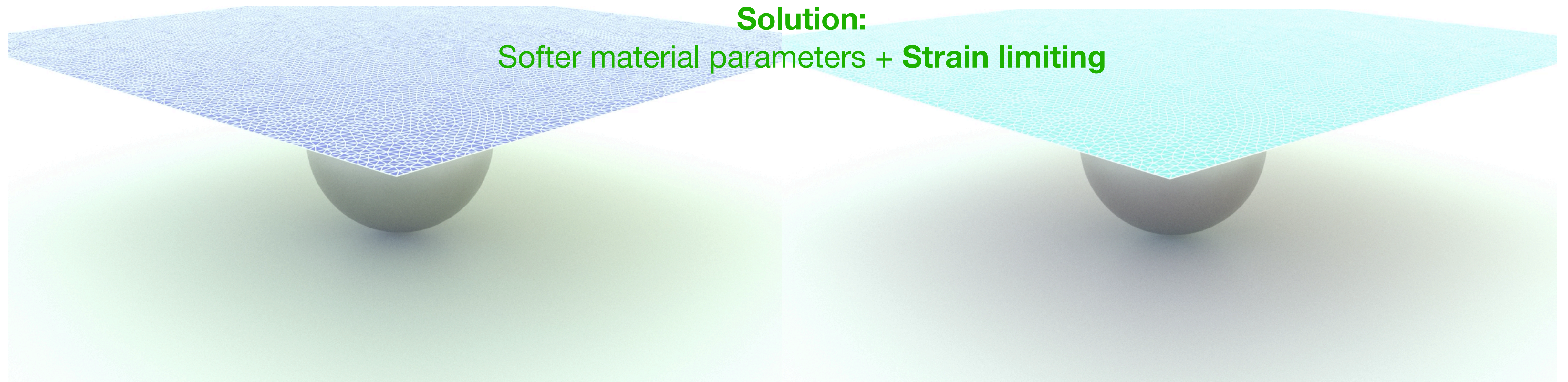


# Simulating Cloth using Surface Meshes

## Membrane Locking

Cloth are nearly unstretchable — stiff stretch resistance,  $E = \sim 10^7$  Pa

With low-res triangulation, there can be geometric lockings:



Stiff membrane creates extra bending penalty  
(real material parameter)

Stretchy artifact  
(0.01x real material parameter)



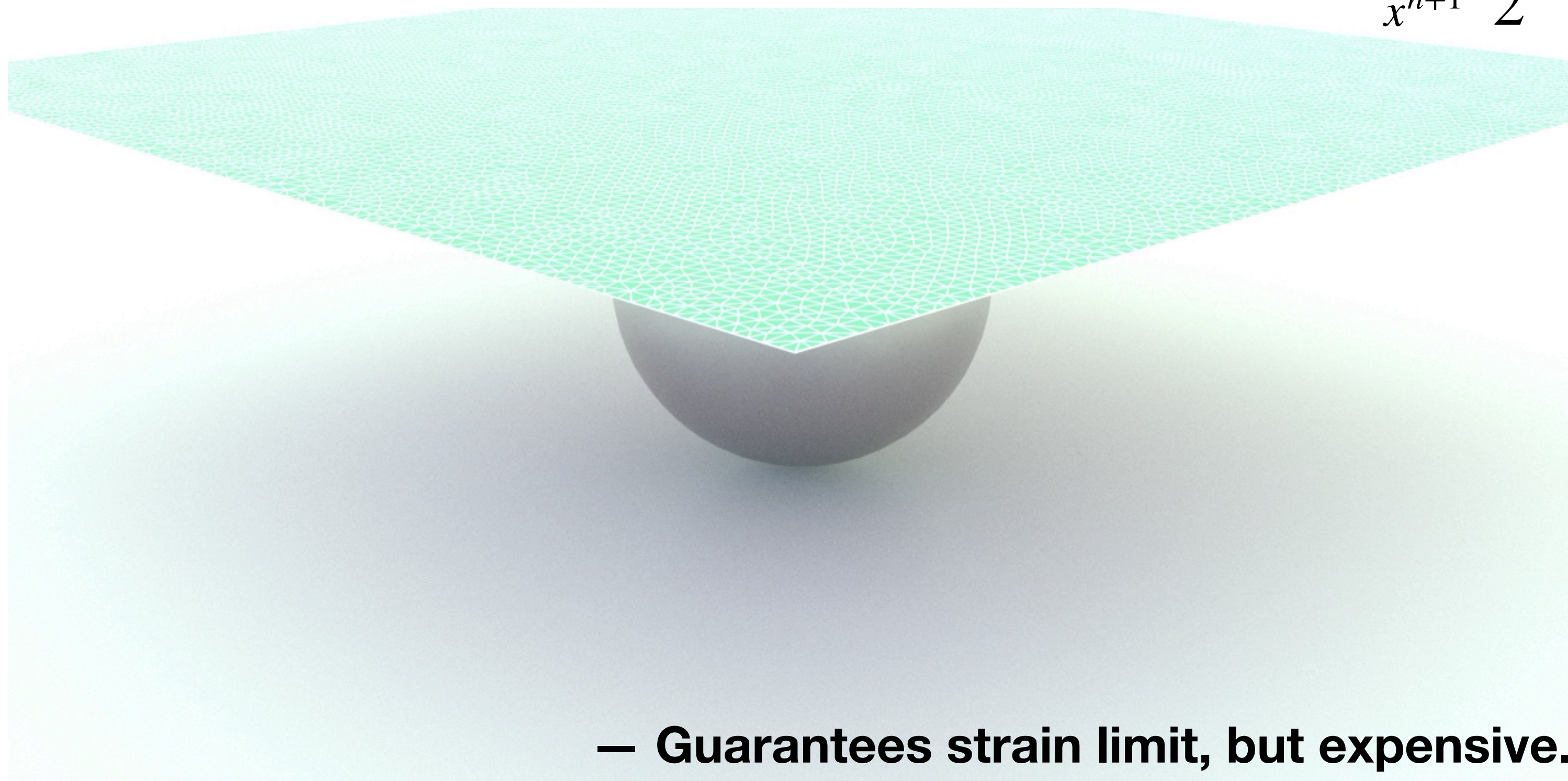
# Simulating Cloth using Surface Meshes

## Avoiding Membrane Locking with Strain Limiting

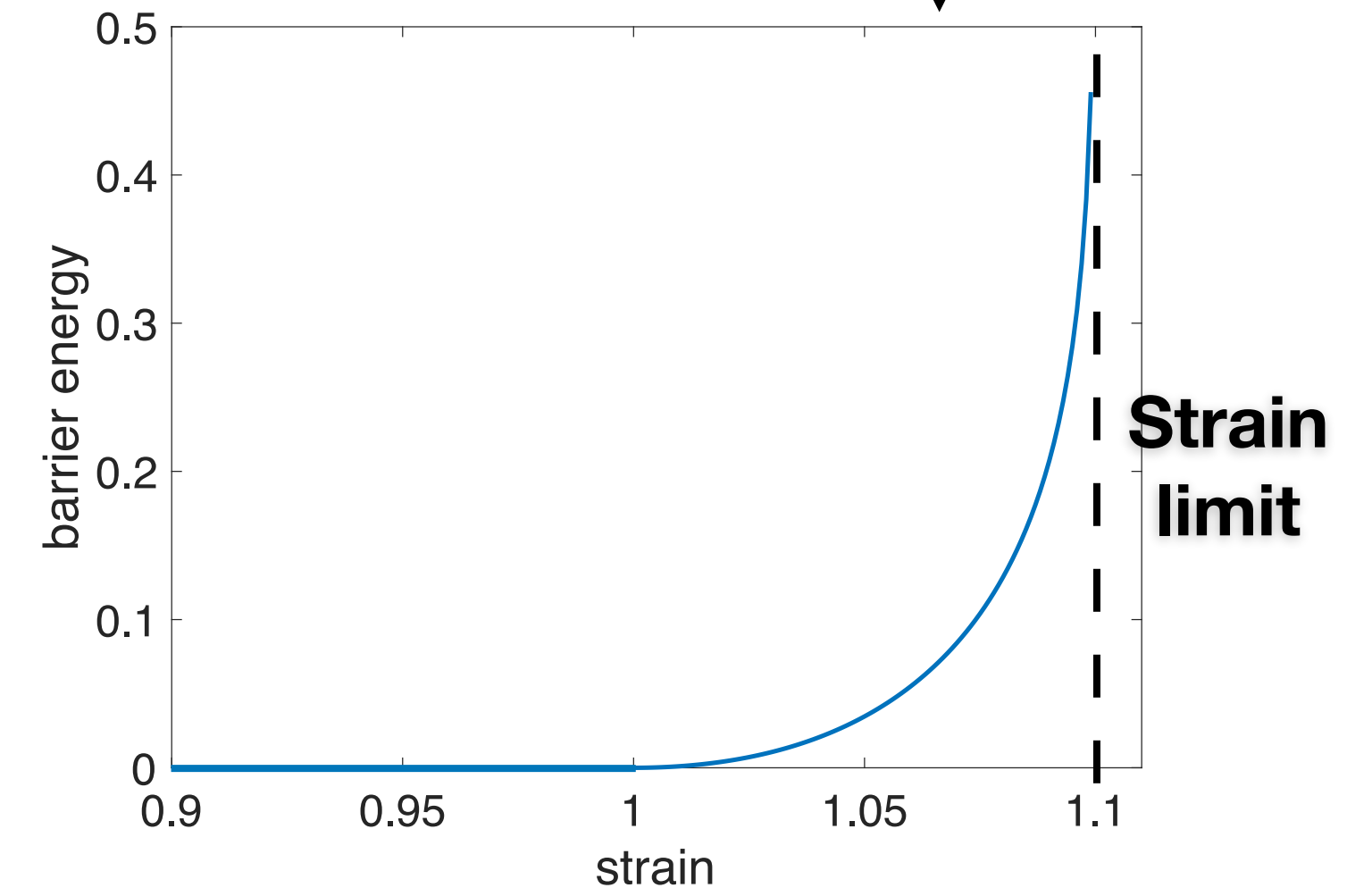
$$\forall t, i \quad S_{t,i} < s \quad (F_t = U_t S_t V_t^T \text{ is the SVD of triangle } t\text{'s deformation gradient})$$

↓  
**Strain limit**

$$\min_{x^{n+1}} \frac{1}{2} \|x^{n+1} - \tilde{x}^n\|_M^2 + h^2(\Psi(x^{n+1}) + B_s(x^{n+1}))$$



$$B_s(x^{n+1}) = \overset{\text{Stiffness}}{\kappa_s} \sum_{t,i} \overset{\text{Volume weighting}}{\mathcal{V}_t} \underset{\text{Barrier function}}{b(S_{t,i}(x^{n+1}))}$$



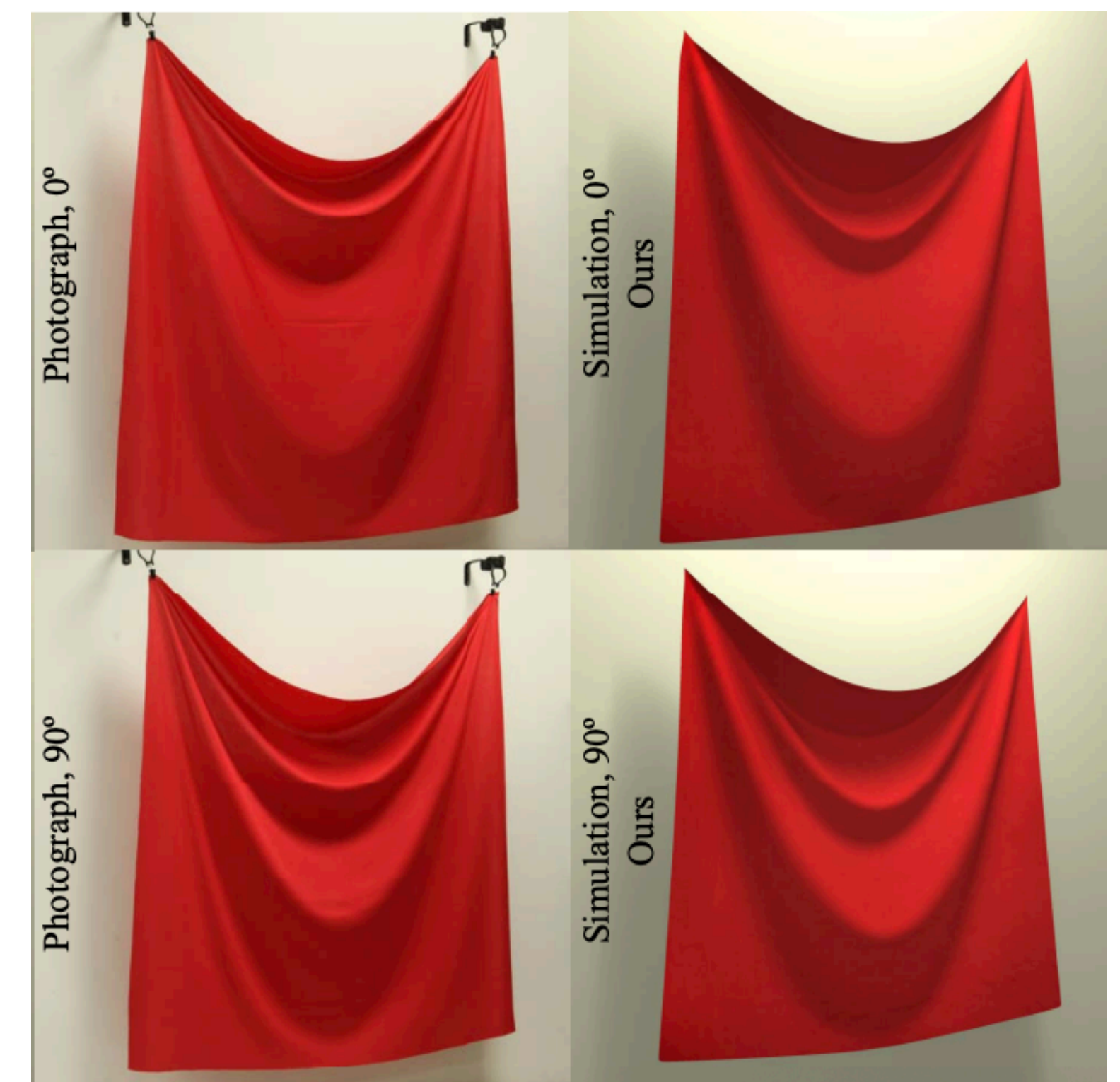
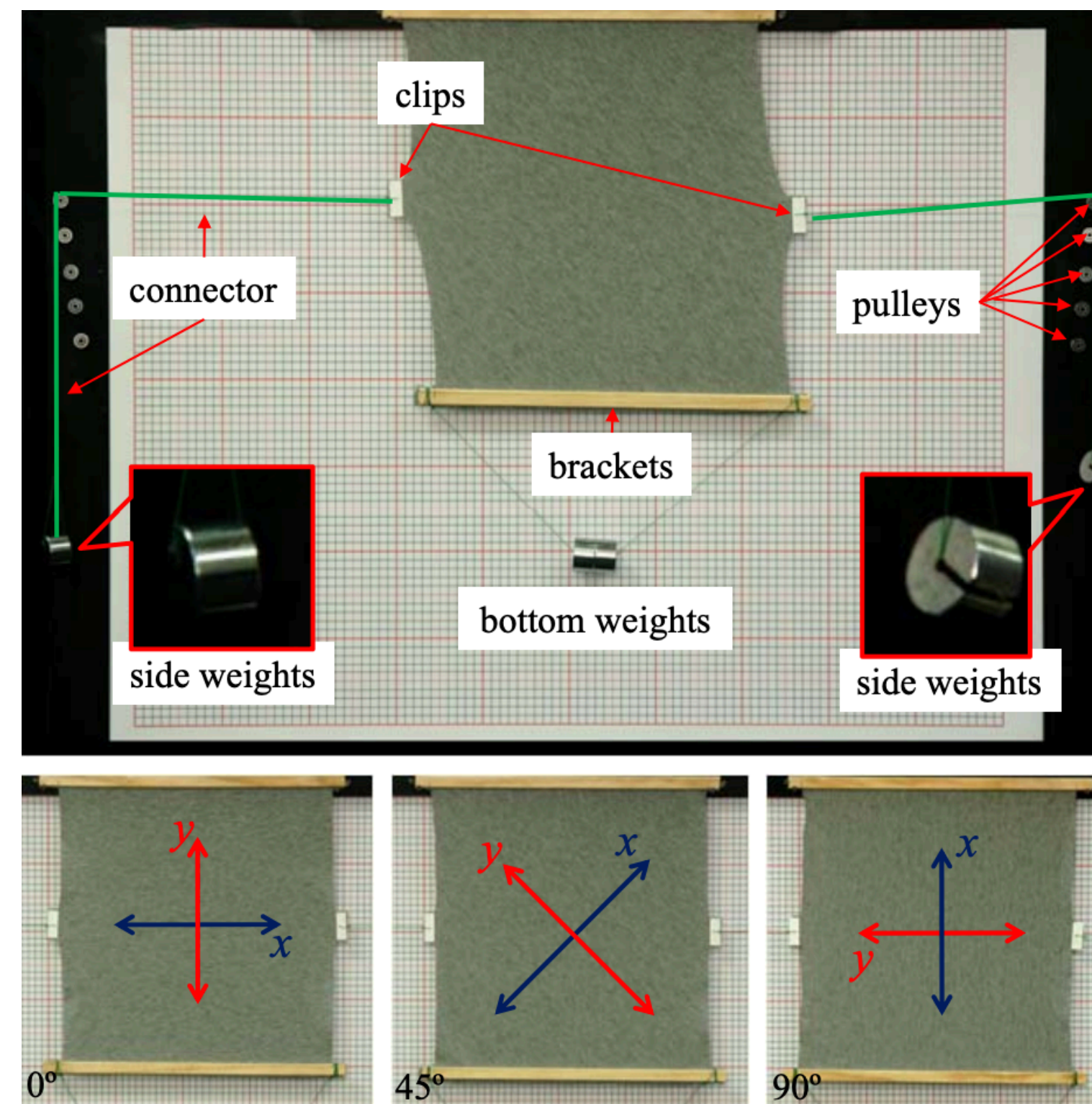
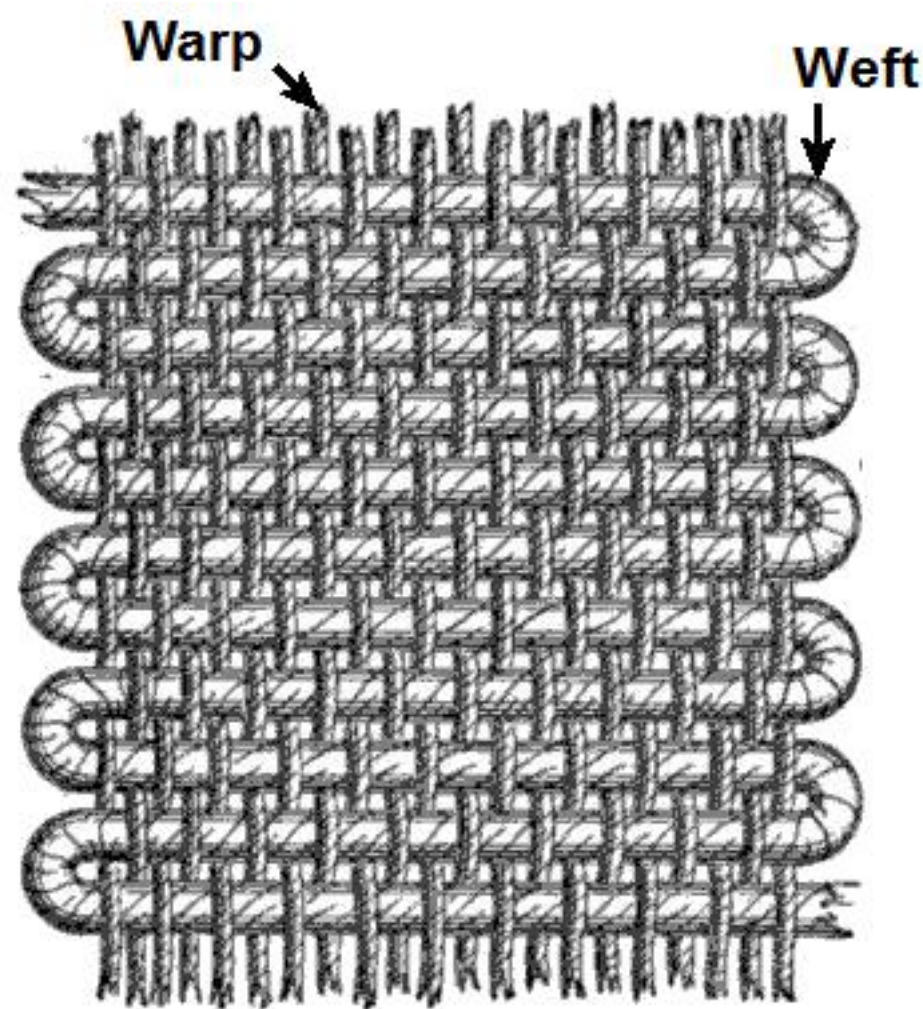


# Simulating Cloth using Surface Meshes

## Anisotropy

Because of the knitting pattern, cloth is anisotropic

Measuring and simulating cloth based on anisotropic elasticity [Wang et al. 2011]:

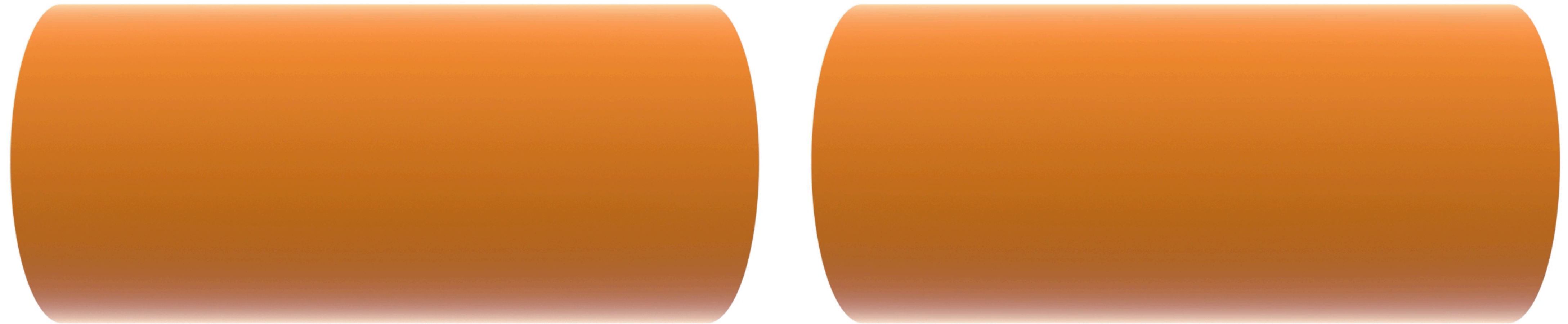




# Simulating Cloth using Surface Meshes

## Thickness Modeling — Inelastic Thickness via Contact Gap

However, there are scenarios where thickness matters:



$$\min_{x^{n+1}} \frac{1}{2} \|x^{n+1} - \tilde{x}^n\|_M^2 + h^2 \sum P(x^{n+1}) + \kappa \sum_k b(D_k(x^{n+1}) - \xi, \hat{d})$$

elastic thickness

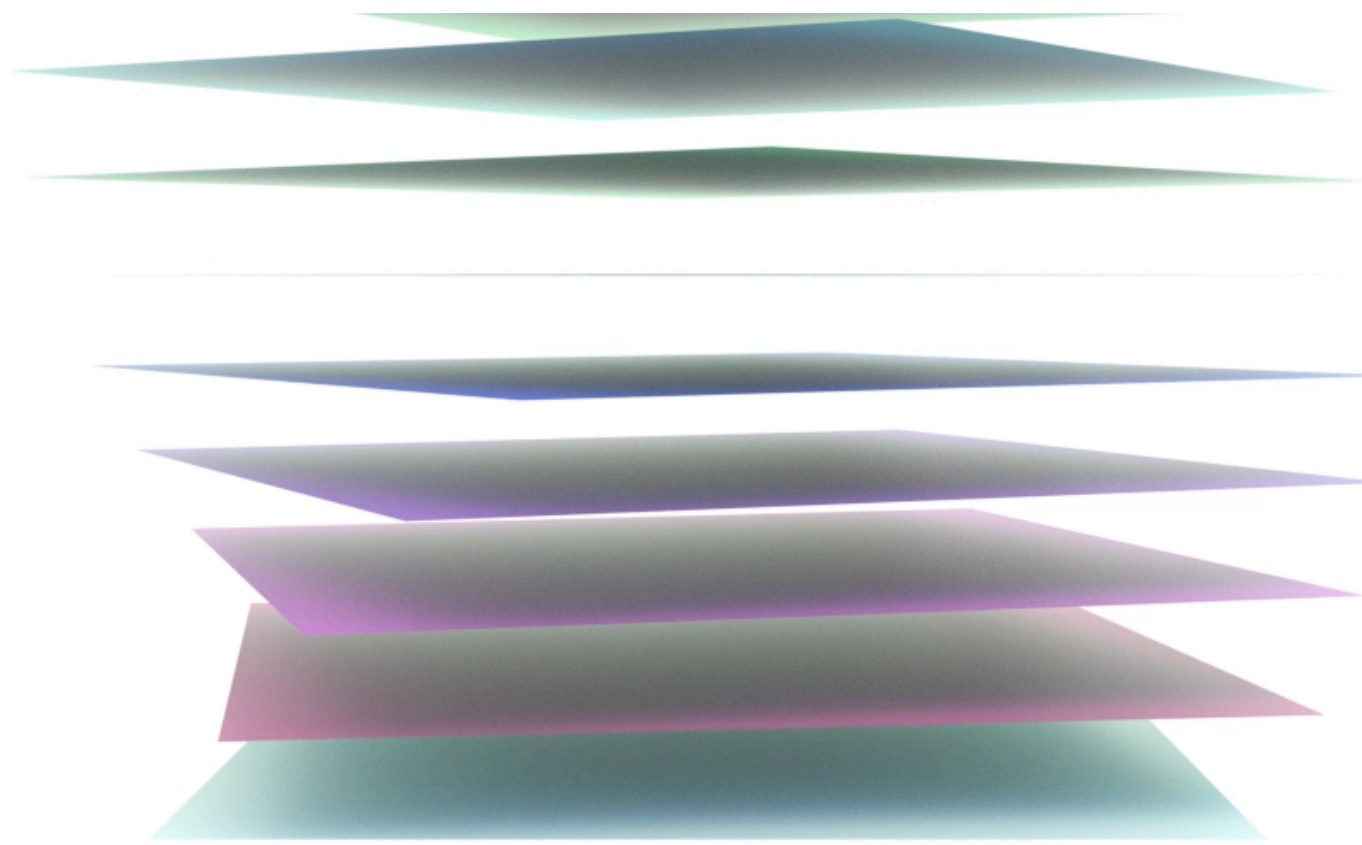
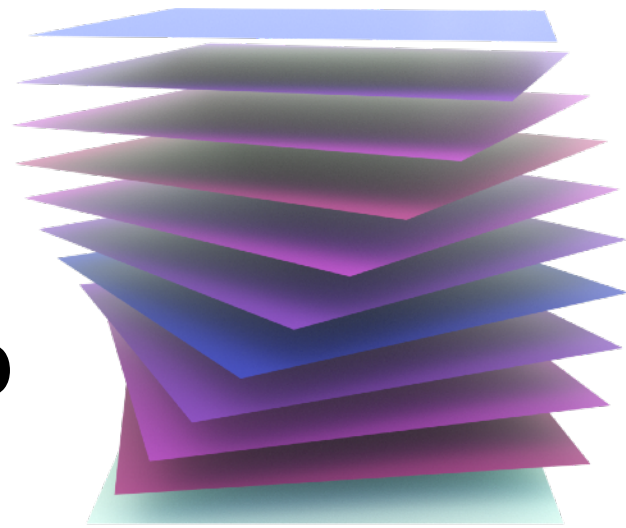
inelastic thickness



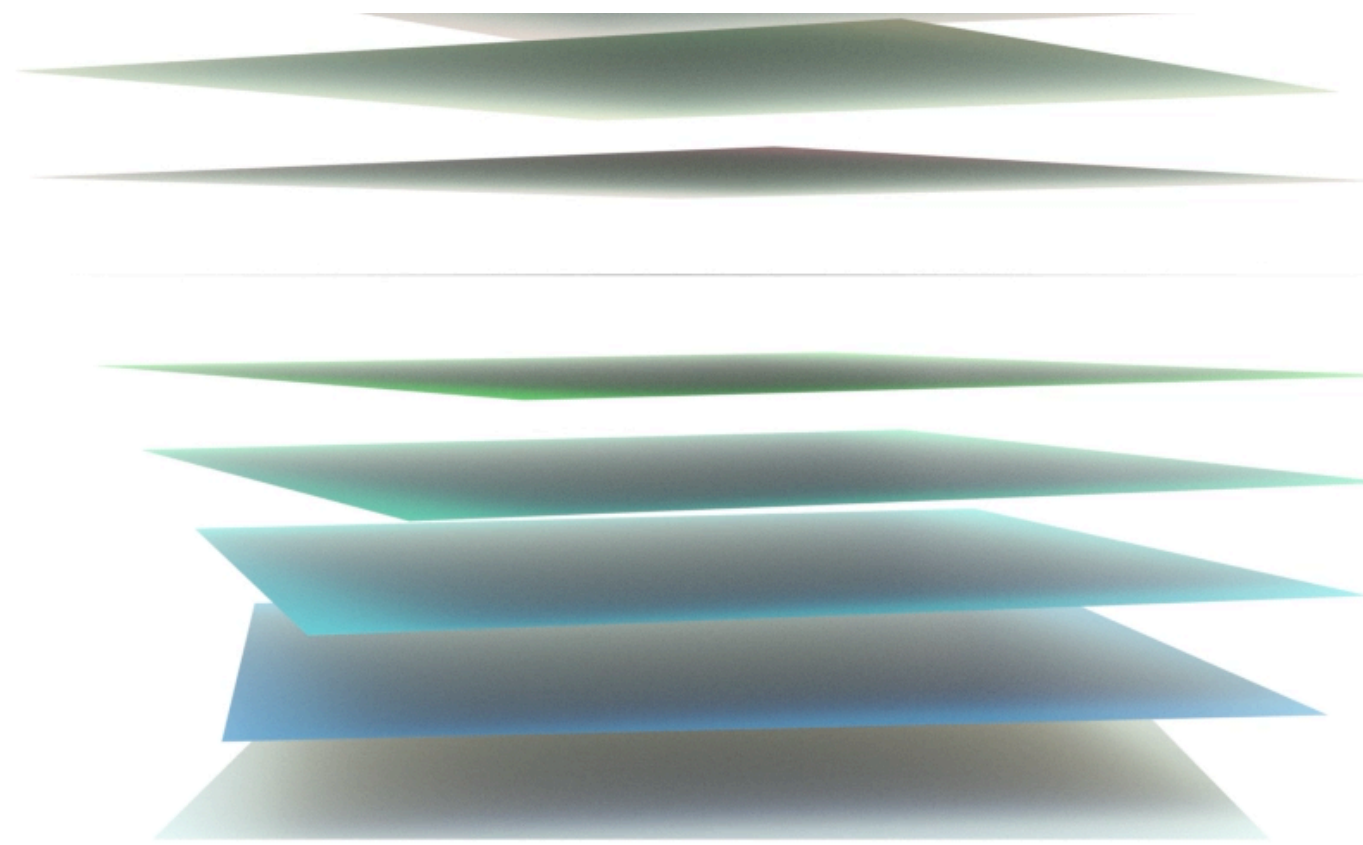
# Simulating Cloth using Surface Meshes

## Thickness Modeling — Elastic Thickness via Barrier $\hat{d}$

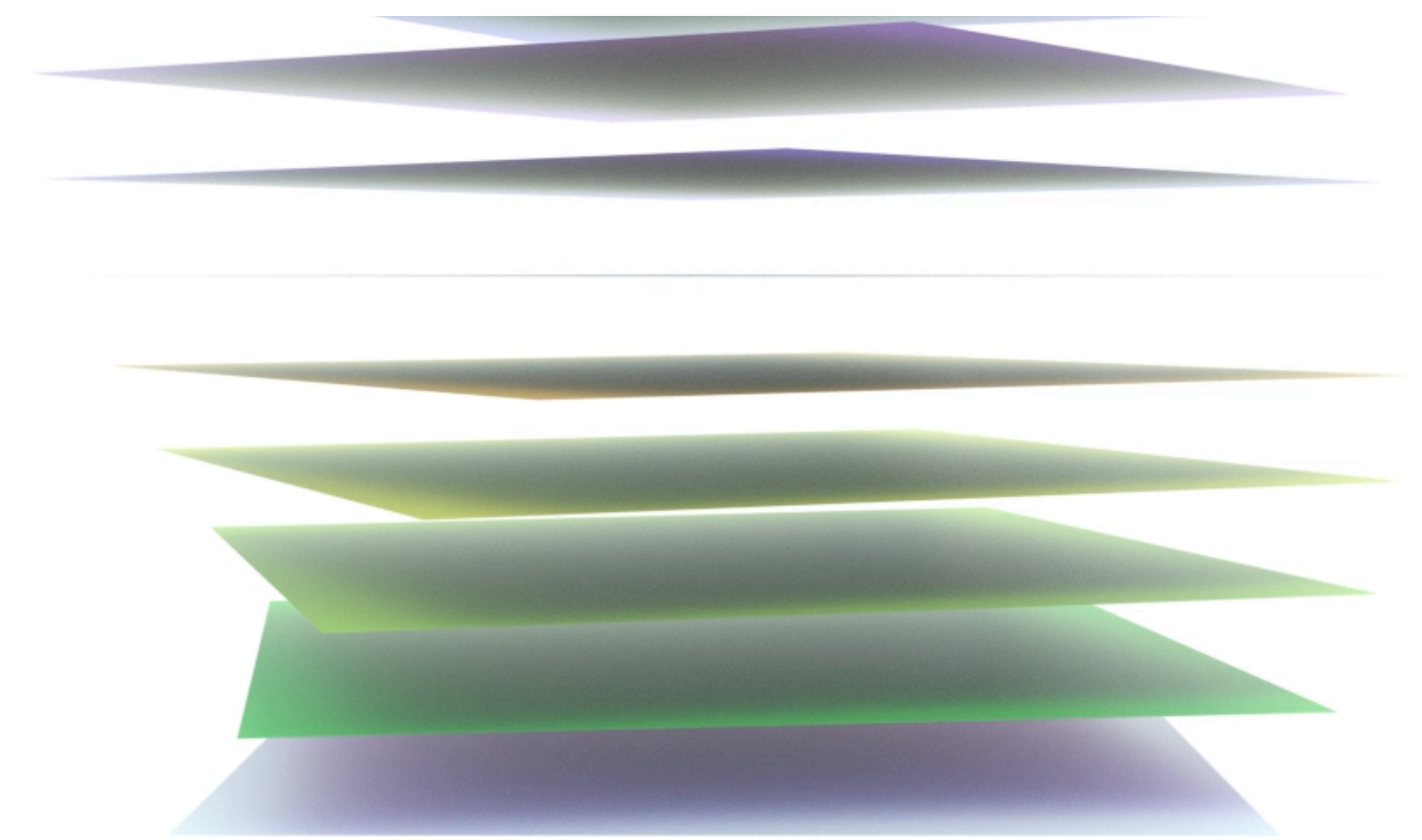
Scene setup



$$\hat{d} = 1mm, h = 0.04s$$



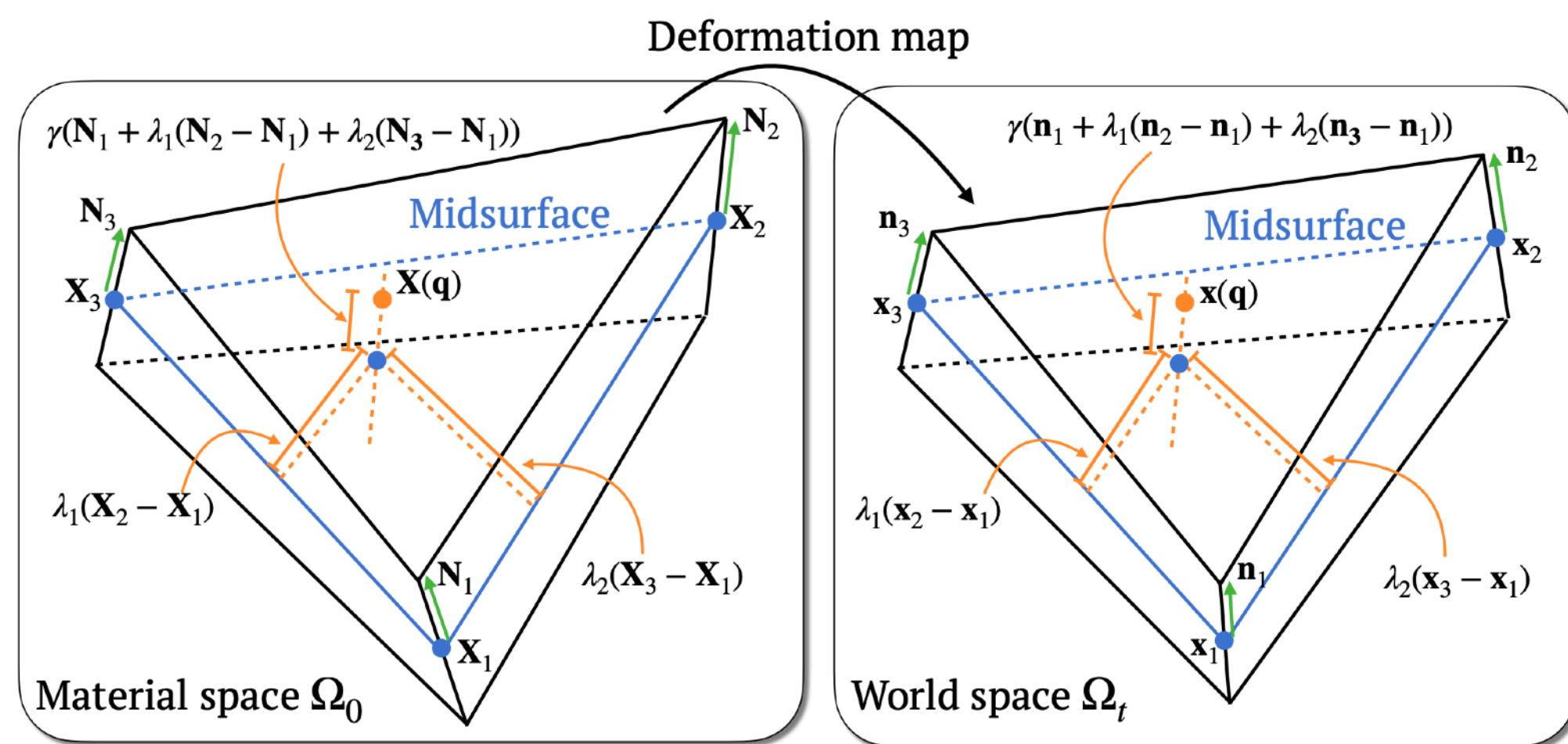
$$\hat{d} = 5mm, h = 0.04s$$



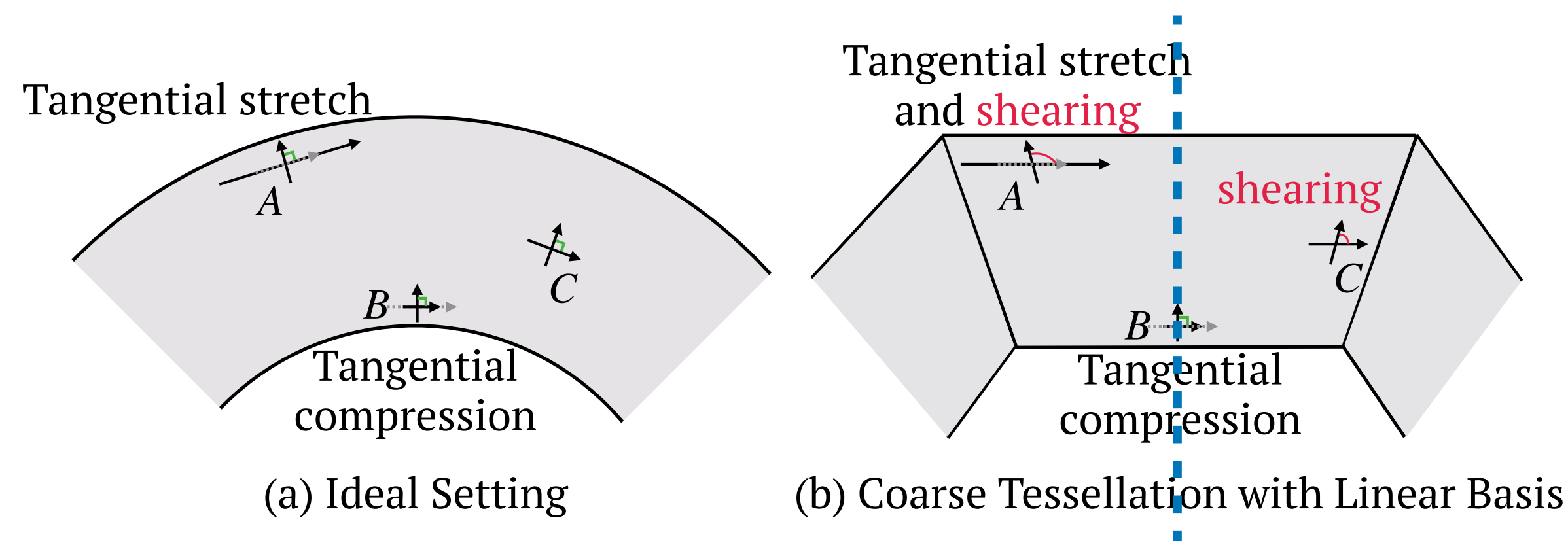
$$\hat{d} = 10mm, h = 0.04s$$



# \*Simulating Thick Shells [Chen et al. 2023]

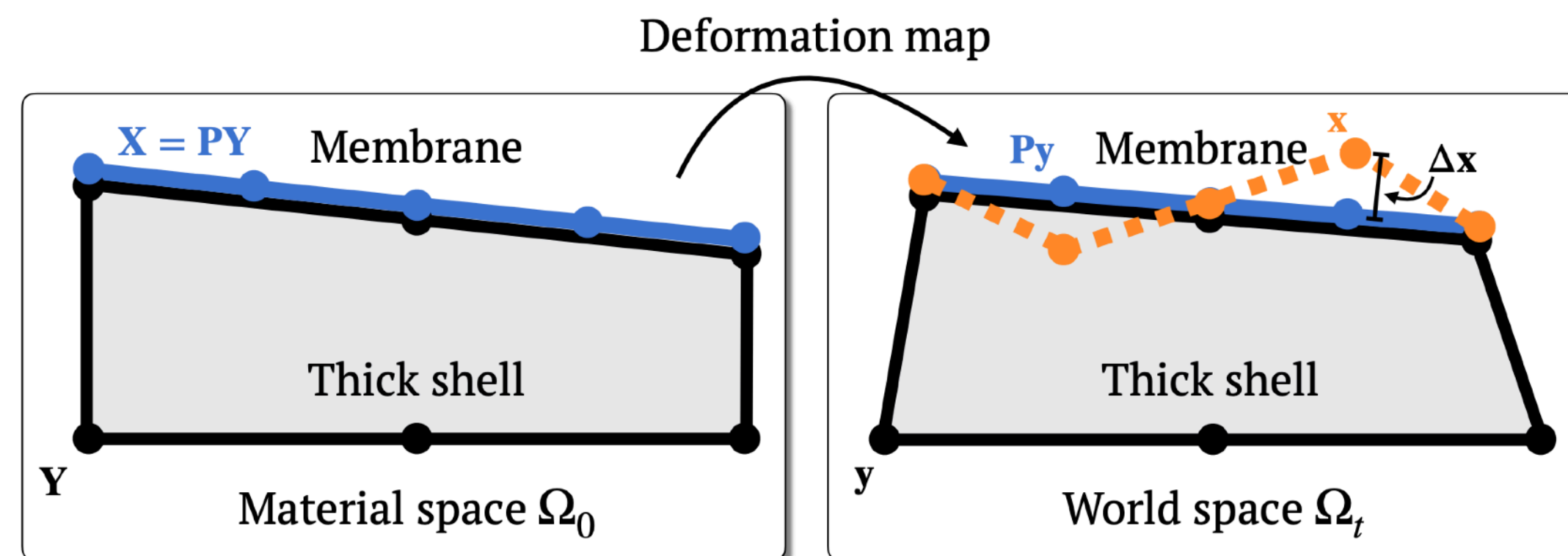


Use prism elements to explicitly track thickness



Place quadratures only on the center line to avoid shear locking

Couple with a fine thin shell to simulate wrinkles





# **\*Simulating Thick Shells [Chen et al. 2023]**

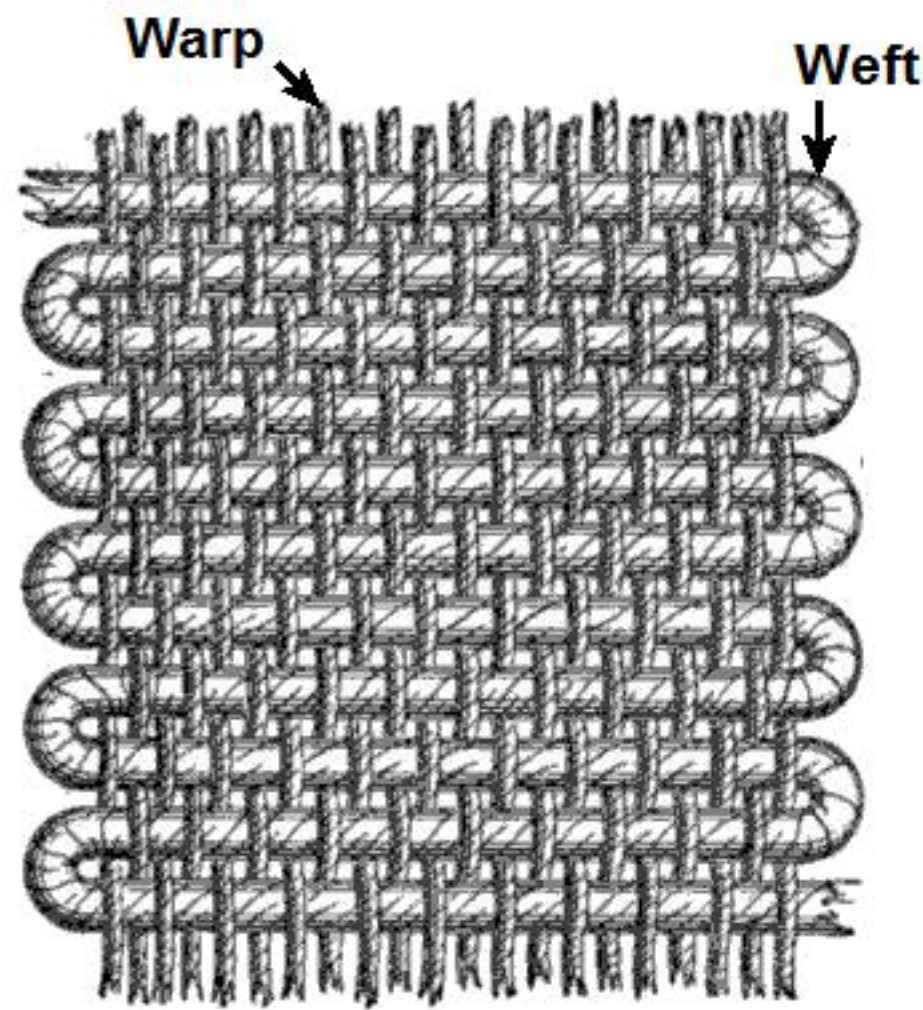


Chen, Yunuo, et al. "Multi-layer thick shells." ACM SIGGRAPH 2023 Conference Proceedings.



# Simulating Yarn-Level Cloth

Because of the knitting pattern, cloth is anisotropic

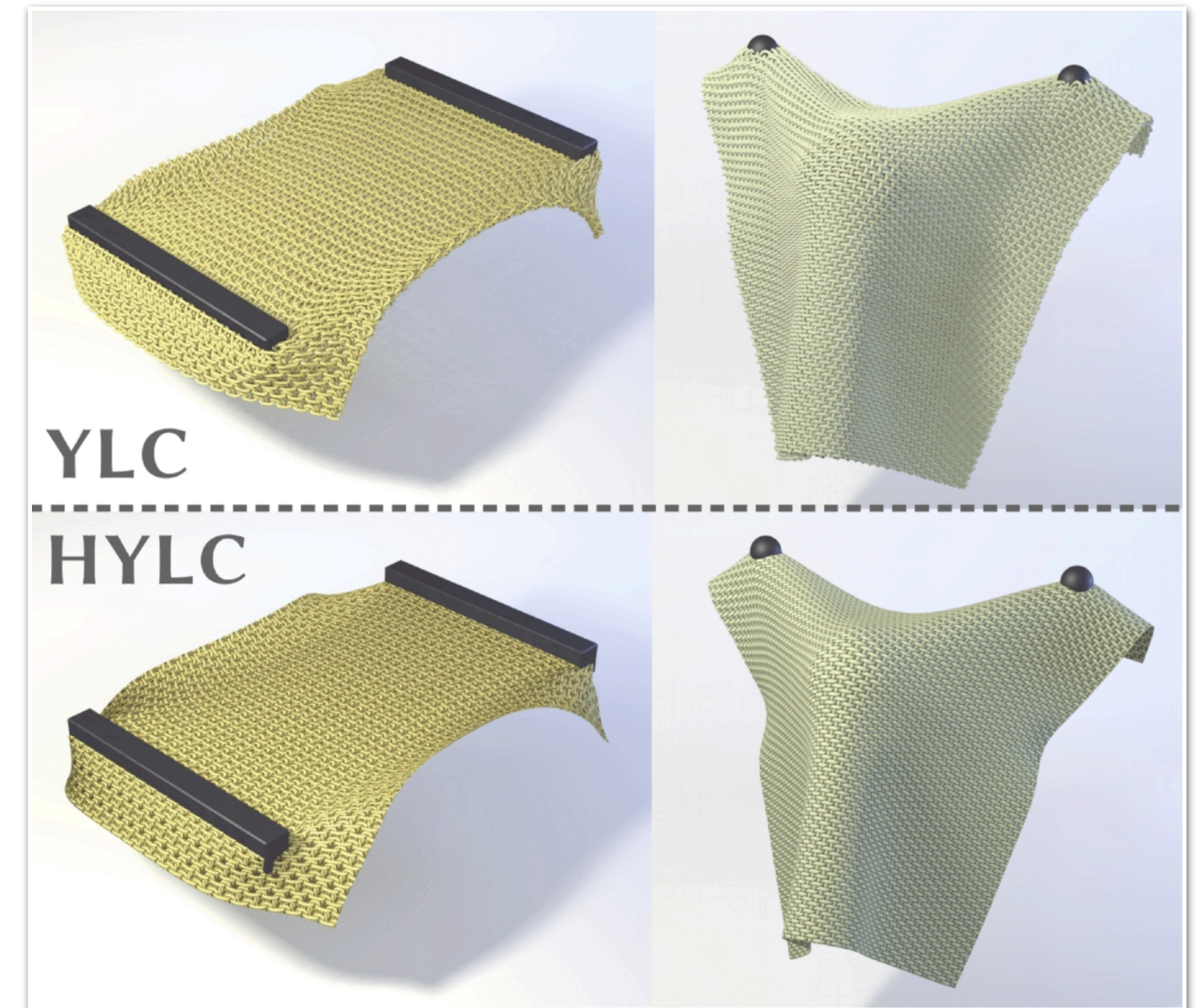


Can directly simulate these yarns!



**Figure 9: Scarf:** Our contact model scales to support the complex contact and folding that occurs in cloth.

[Kaldor et al. 2008]



[Sperl et al. 2020]



# Topics Today:

- Thin Shells
  - ▶ Stretching and Bending
  - ▶ Strain Limiting and Thickness
- **Rods and Particles**



# Simulating Slender Rods using Polylines

## Discrete Elastic Rods

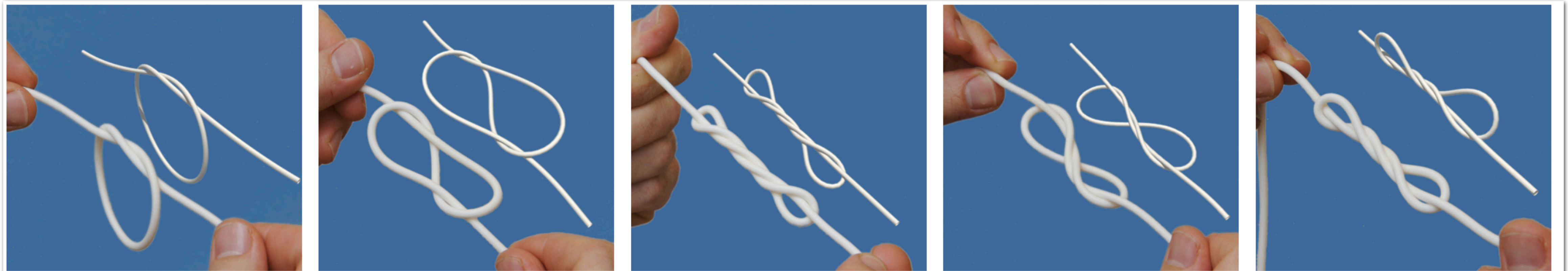


Figure 1: **Experiment and simulation:** A simple (trefoil) knot tied on an elastic rope can be turned into a number of fascinating shapes when twisted. Starting with a twist-free knot (*left*), we observe both continuous and discontinuous changes in the shape, for both directions of twist. Using our model of Discrete Elastic Rods, we are able to reproduce experiments with high accuracy.

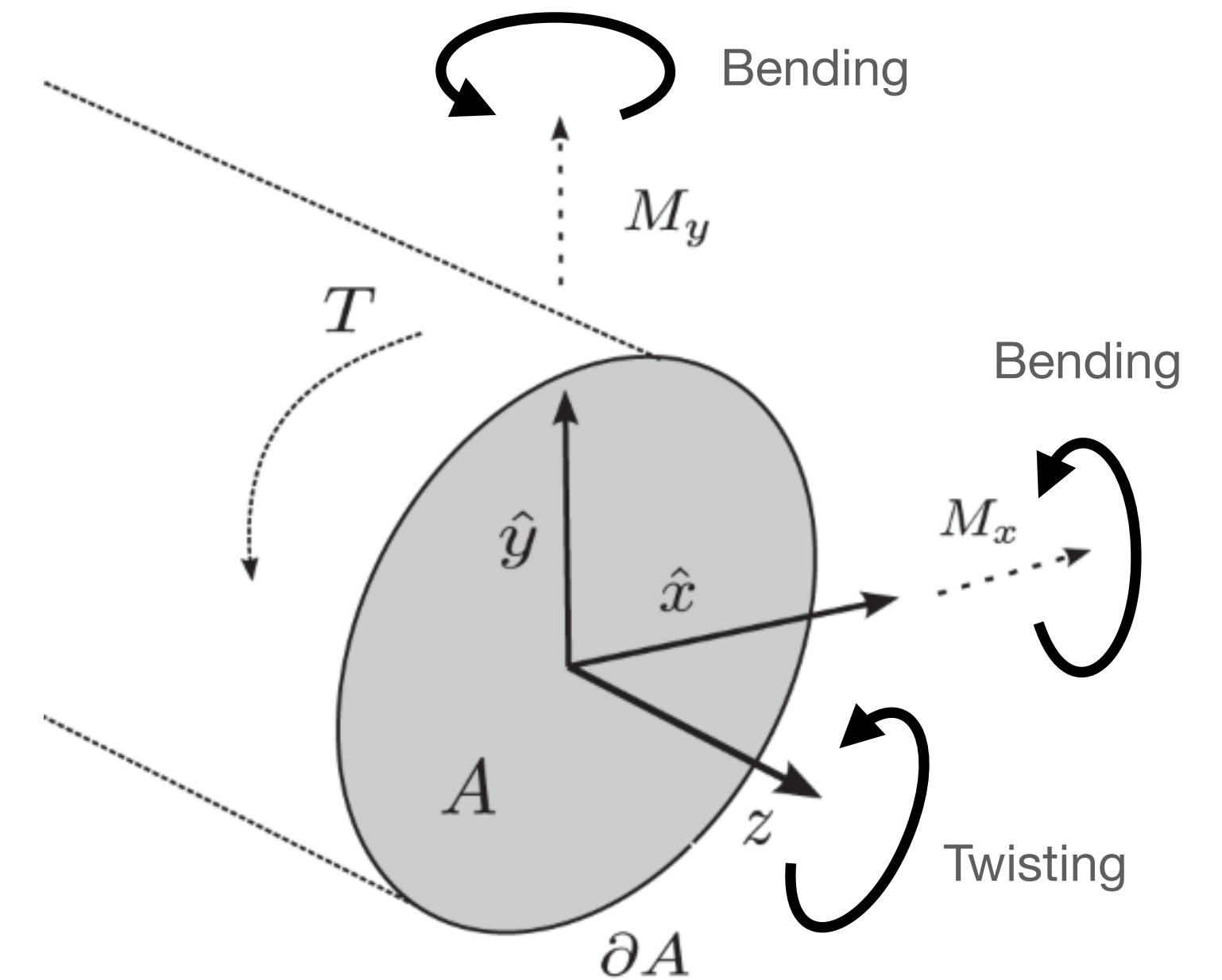
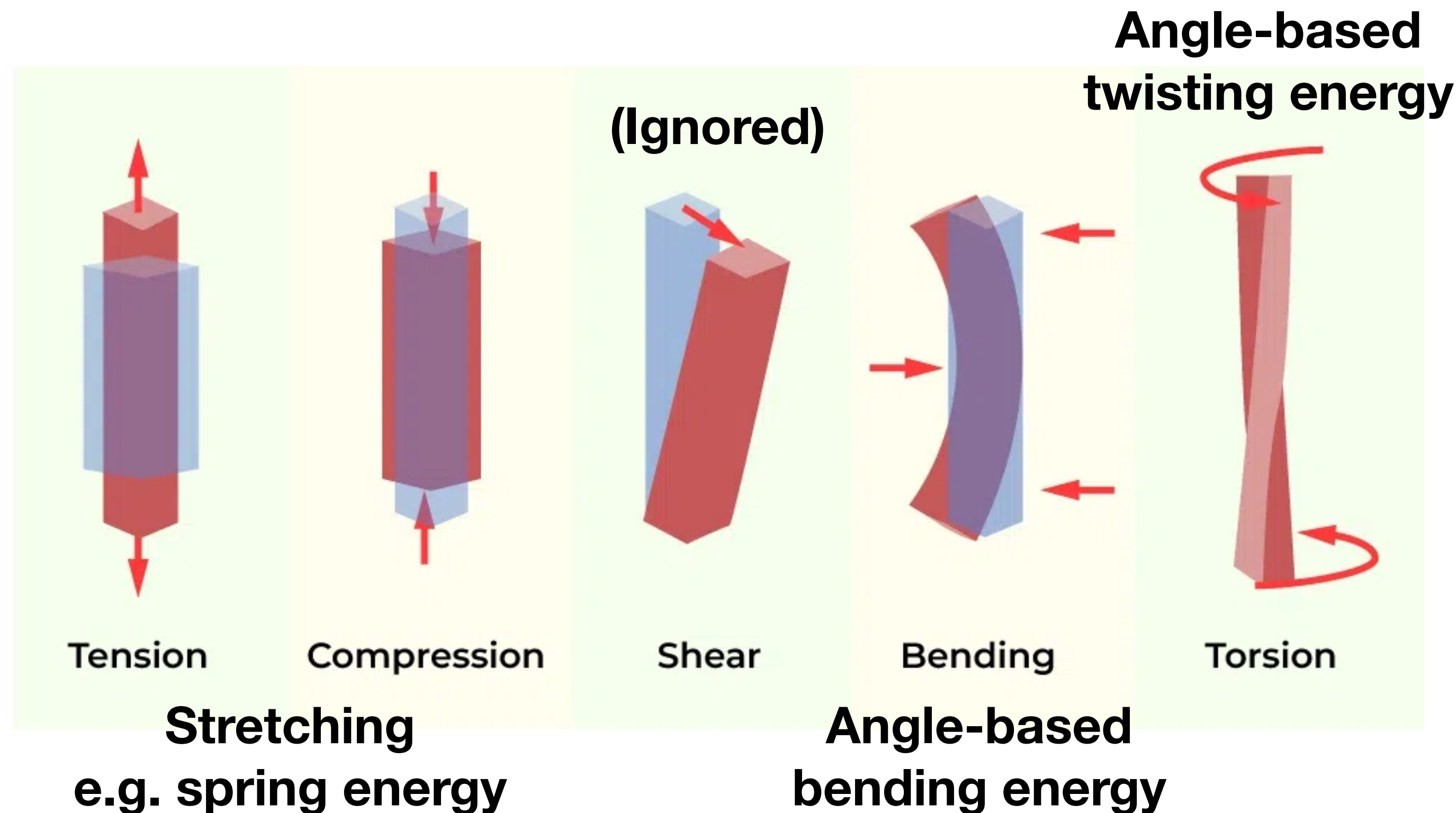
[Bergou et al. 2008]

Strain energy decomposed into **stretching**, **bending**, and **twisting** energies.

# Simulating Slender Rods using Polylines

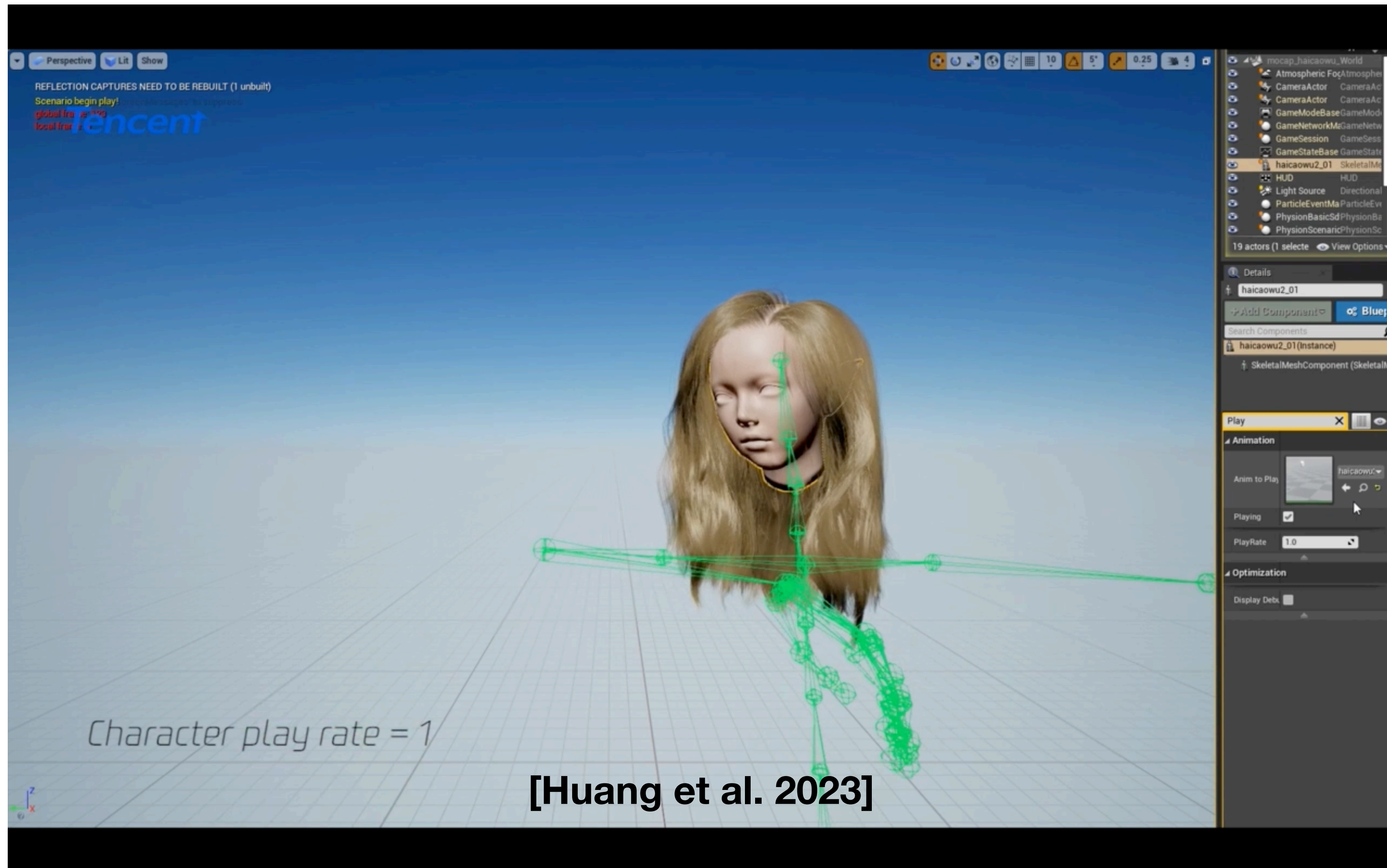
## Discrete Elastic Rods — Stretching, Bending, Twisting

- Idea: use rotation angles of local frames to measure bending and twisting deformation:



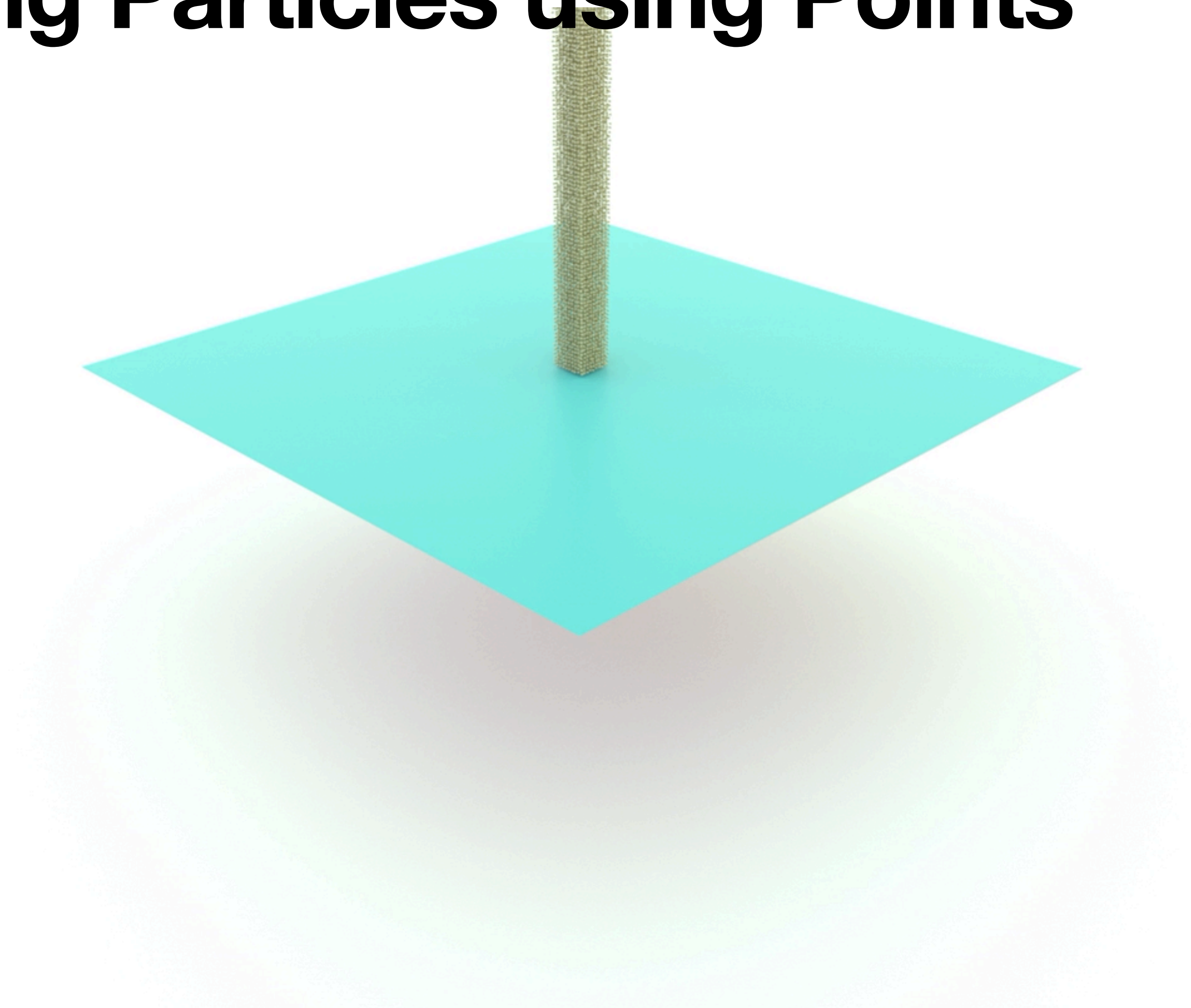


# Simulating Hairs using Polylines



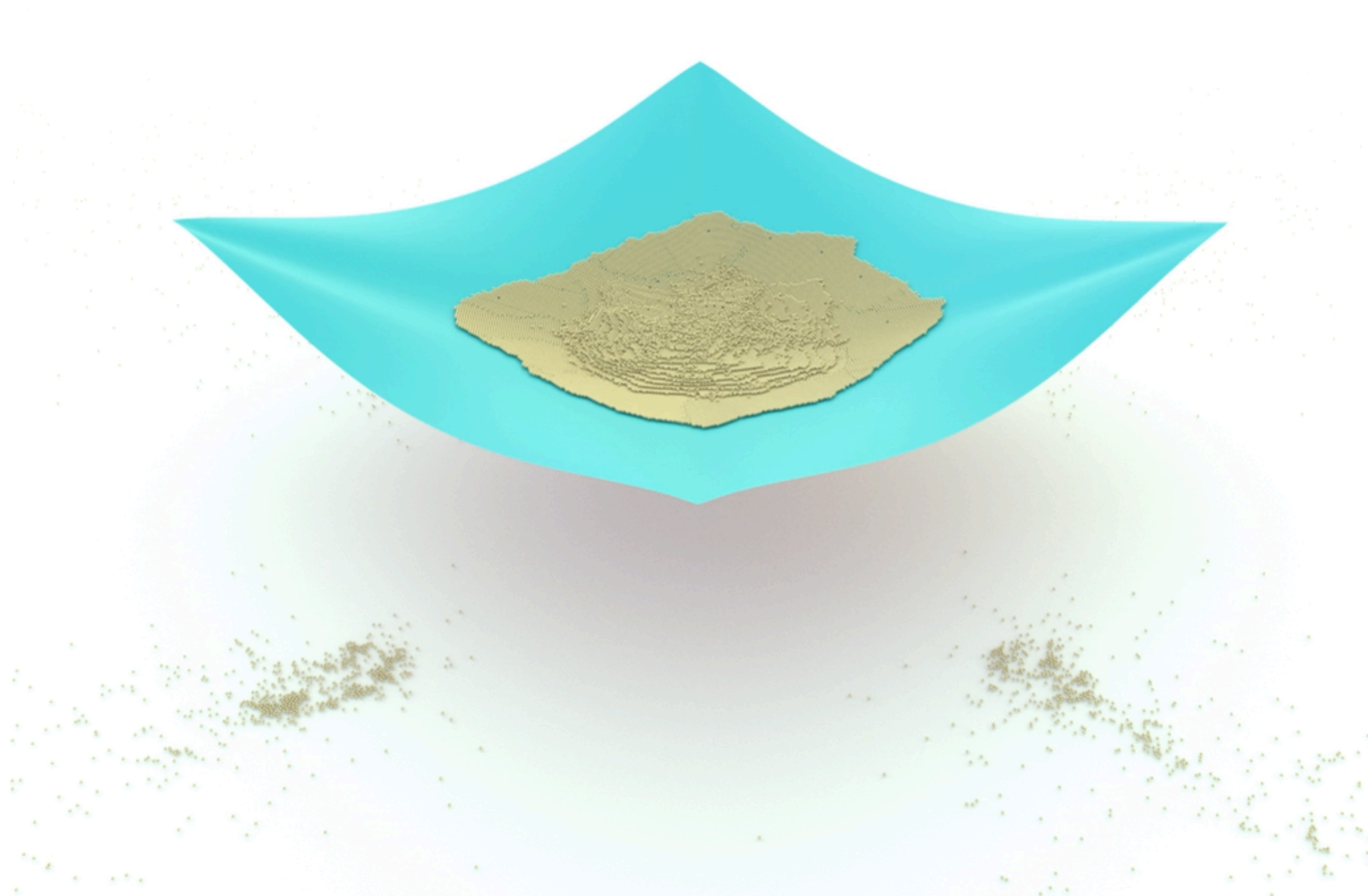
[Huang et al. 2023]

# Simulating Particles using Points



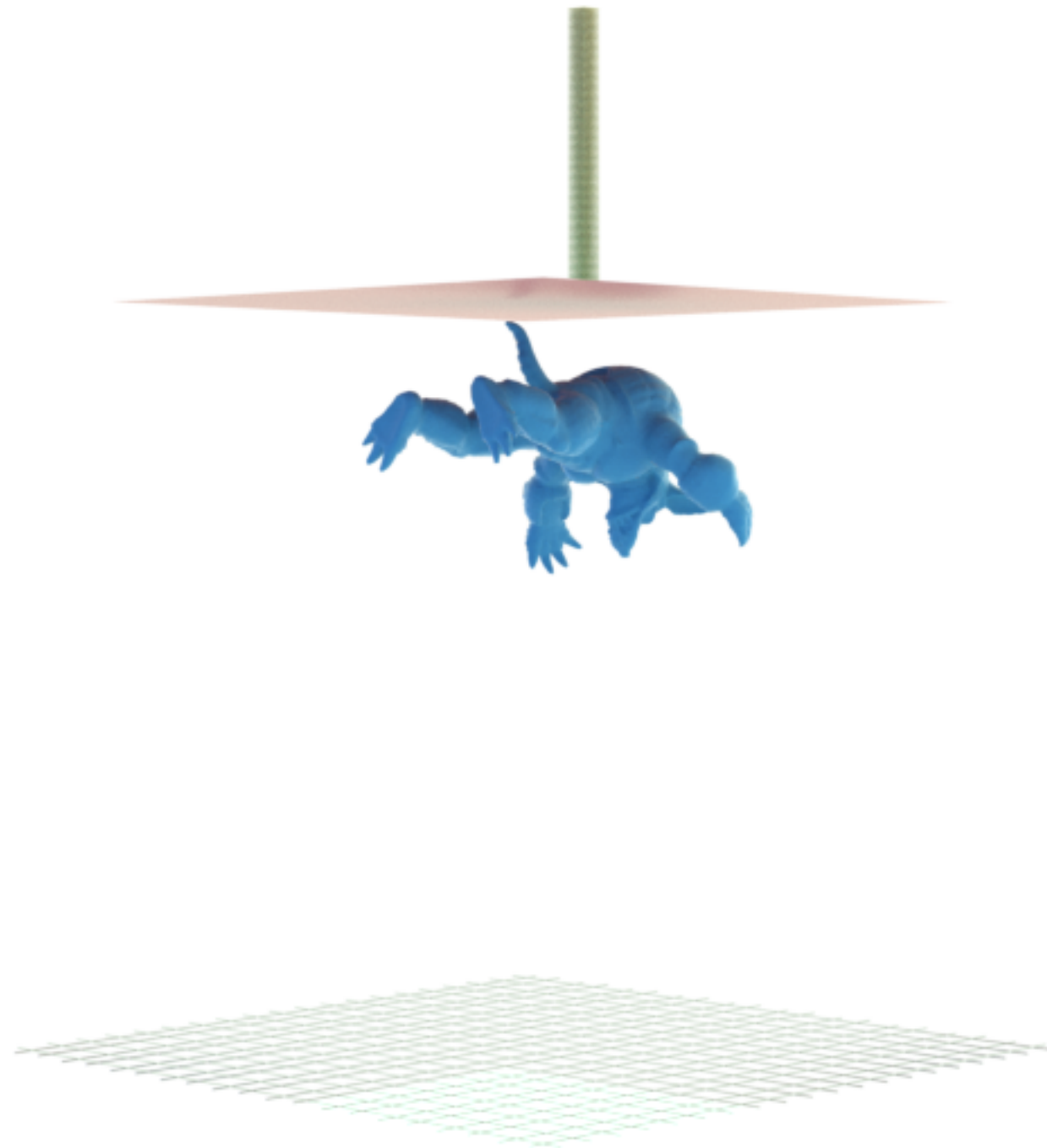


# Simulating Particles using Points

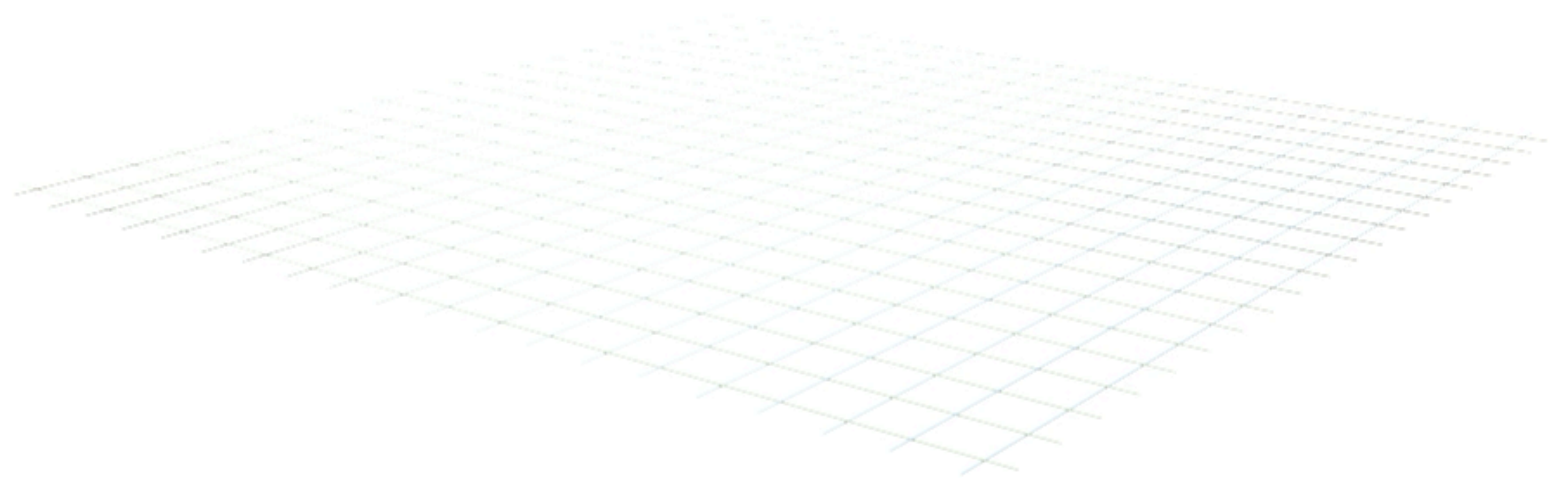


# “All-In”

## Coupled using IPC



**Scene setup**

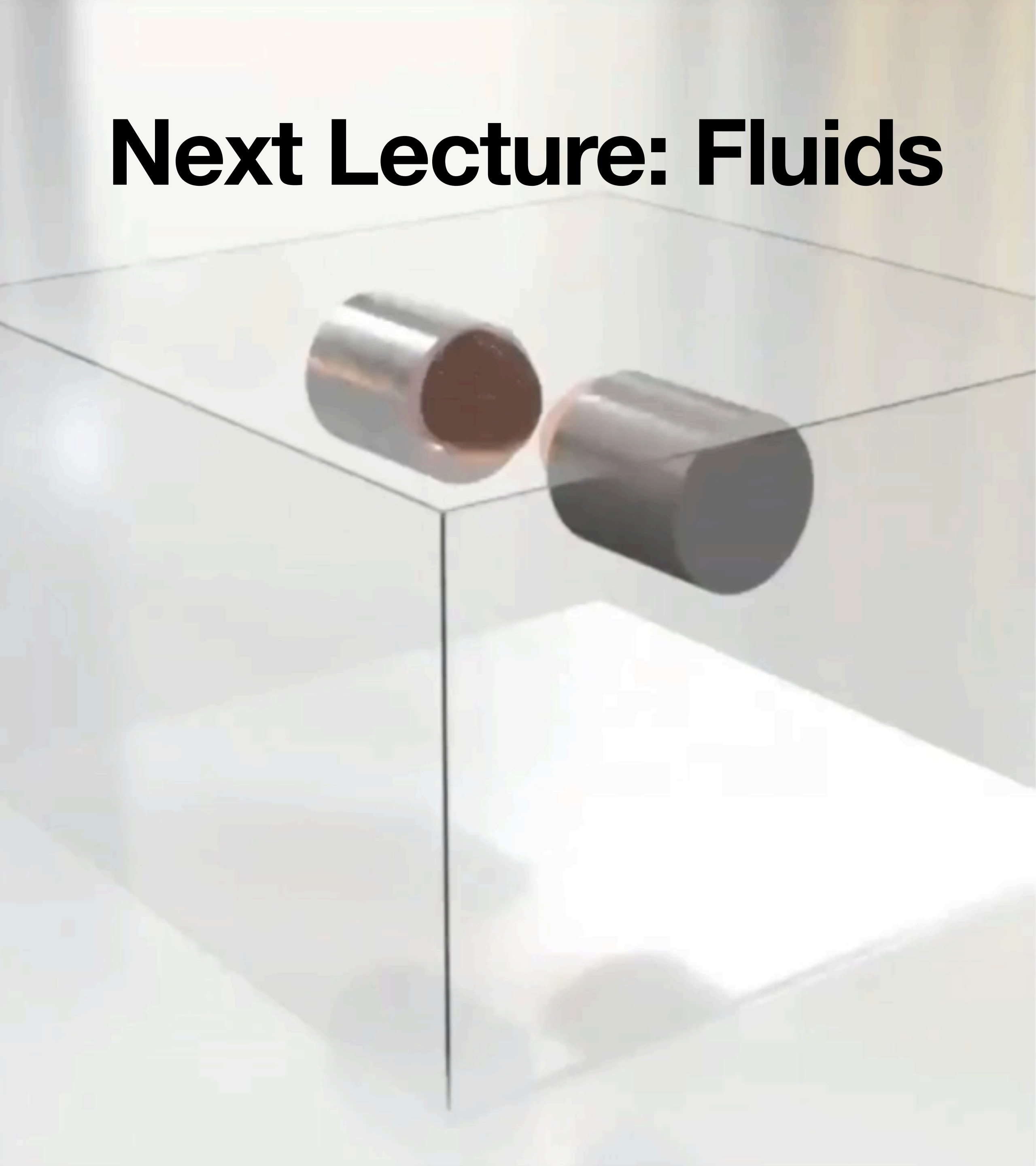


# Topics Today:

- **Thin Shells**
  - ▶ **Stretching and Bending**
  - ▶ **Strain Limiting and Thickness**
- **Rods and Particles**



# Next Lecture: Fluids



# Image Sources

- <http://viterbi-web.usc.edu/~jbarbic/femdefo/barbic-courseNotes-modelReduction.pdf>
- [https://www.researchgate.net/publication/332914440\\_Port-Hamiltonian formulation and symplectic discretization of plate models Part II Kirchhoff model for thin plates/figures?lo=1&utm\\_source=google&utm\\_medium=organic](https://www.researchgate.net/publication/332914440_Port-Hamiltonian_formulation_and_symplectic_discretization_of_plate_models_Part_II_Kirchhoff_model_for_thin_plates/figures?lo=1&utm_source=google&utm_medium=organic)
- <http://multires.caltech.edu/pubs/ds.pdf>
- <https://cims.nyu.edu/gcl/papers/bergou2006qbm.pdf>
- [https://en.wikipedia.org/wiki/Warp\\_and\\_weft](https://en.wikipedia.org/wiki/Warp_and_weft)
- <https://wanghmin.github.io/Wang-2011-DEM/Wang-2011-DEM.pdf>
- [https://www.cs.cornell.edu/projects/YarnCloth/sg08\\_knityarns.pdf](https://www.cs.cornell.edu/projects/YarnCloth/sg08_knityarns.pdf)
- <https://visualcomputing.ist.ac.at/publications/2020/HYLC/>
- <https://www.cs.columbia.edu/cg/pdfs/143-rods.pdf>
- <https://www.geeksforgeeks.org/stress-and-strain/>
- <https://www.nature.com/articles/s41598-019-52878-z>
- <https://www.youtube.com/watch?v=UDQaw4Ff3sg>